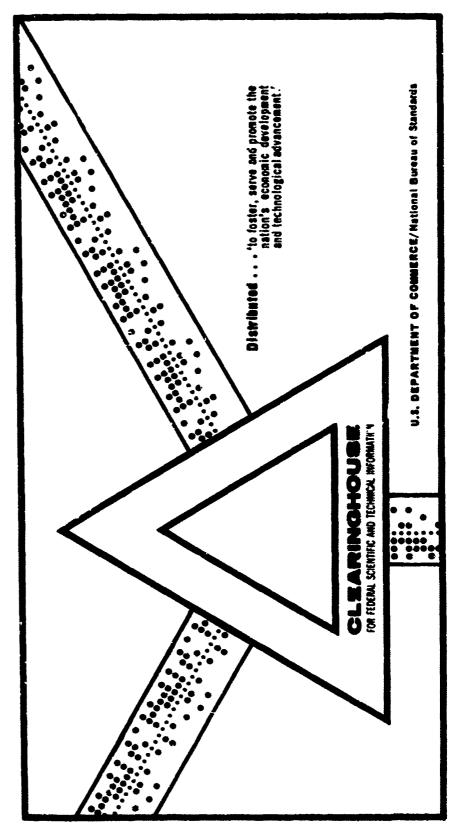
PROTOTYPE CLUSTER PARACHUTE RECOVERY SYSTEM FOR A 50,000-POUND UNIT LOAD. VOLUME II. DIRECT DESIGN ASPECTS

Royce A. Tout, et al

Pioneer Paraciute Company, Incorporated Manchester, Connection!

January 1969



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TECHNICAL REPORT
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PROTOTYPE CLUSTER-PARACHUTE RECOVERY SYSTEM
FOR A 50,000-1b UNIT LOAD
VOLUME II- DIRECT DESIGN ASPECTS

by

Royce A. Toni and Milan M. Kror

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Contract No. DAAG17-68-0142

Project Reference: 1F162203D195

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Airdrop Engineering Laboratory U. S. ARMY NATICK LABORATORIES Natick, Massachusetts 01760

FOREWORD

This work was initiated in an effort toward the design and fabrication of a prototype recovery parachute assembly to enable the airdrop, by use of parachutes in a cluster, of a 50,000-1b unit load. The first phase of this study was concerned solely with the design aspects; the second phase deal; with fabrication.

Volume II presents the results of work on the direct design of the selected prototype parachute assembly.

This work was conducted under U. S. Army Project 1F162203D-195, Exploratory Development of Airdrop Systems, by Pioneer Parachute Company, Manchester, Connecticut, under contract DAAG17-68-0142.

The project engineer was Mr. Royce A. Toni of the contracting agency. The work was performed under the direction of Mr. Arthur W. Claridge, the project engineer for the U. S. Army Natick Laboratories.

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ABSTRACT

This report covers the direct design aspects of the selected prototype cargo recovery assembly for airdropping heavy unit loads in the order of 50,000 pounds.

The detailed design of the components is covered as well as stress analysis to determine the margins of safety for the materials selected. Material lists and weights for the components are provided. Laboratory testing of individual components and strength efficiency of stitch patterns are shown.

1. INTRODUCTION

The prototype parachute assembly shown in Fig. 1 represents a system that, when used in a cluster of six, will enable the airdrop of a 50,000-1b unit load. This particular configuration was ultimately selected on the basis of findings presented in Volume I of this series.

As a result of the Volume I study, these aspects deal primarily with detail. The purpose of Volume II is to present the results associated with the direct design aspects.

2. SPECIFICATIONS

a. Design

The parachute system was designed to meet all the requirements stated in Volume I, Section 3.

(1) Parachute Size

The procedure followed to determine the size of the parachute system (135-ft-D) is presented in Volume I, Section 9.

(2) Opening Force

The method used to define the maximum force (28,300 lb) experienced by any one parachute when used in a cluster of six to airdrop a 50,000-1b unit load, is presented in Volume I, Section 10.

Components

The selected prototype assembly consists of the following components:

- (a) canopy, which has the subcomponents
 - (1) suspension lines,
 - (2) vent ring, and(3) center line;
- (b) risers, which have the subcomponent
 - (1) separable link;
- (c) riser extension;
- (d) suspension clevis, which has the subcomponent
 - (1) bushing;

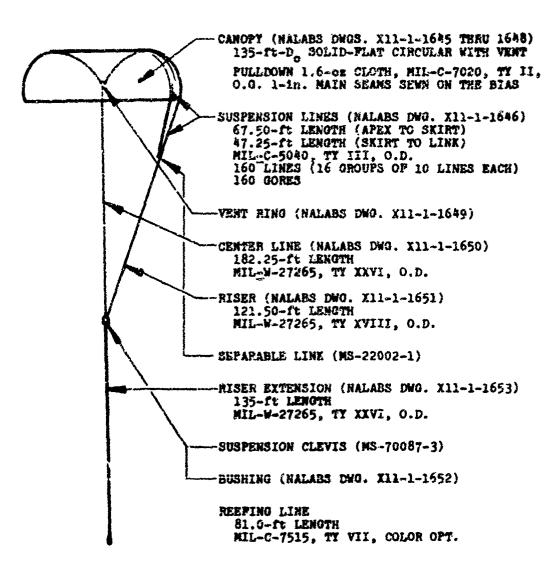


Fig. 1. Prototype assembly selected for use in a cluster of six to airdrop a 50,000-1b unit load.

- (e) deployment bag and bridle; and
- (f) reefing components.

The canopy is solid-flat circular, having a 135-ft nominal diameter (D). The suspension lines are approximately $0.35 \times D$ in length, measured from the separable link to the canopy skirt; the lines continue to run through the canopy main seam up to the vent ring and return to the separable link via the adjacent gore.

The center line is of two-ply construction and has a length approximately equal to the total length of the suspension line and its riser plus $D_{\rm o}/10$.

The risers are approximately 0.65 × D in length. At one end are separable links to which the suspension lines can be tied; the other end is looped so as to be accommodated by the clevis. The construction of this item is such that for every four ends connecting to a total of forty suspension lines (10 lines per end) the other (and only) end connects to the clevis.

The riser extension is of 6-ply construction and looped at one end so as to be fitted into the clevis. The other end, the load-attachment point, is looped to accommodate a hardware fitting supplied by Natick Laboratories.

c. Deployment Conditions and Weight

The gross rigged weight for a cluster of six of the specified parachute assemblies is 50,000 lbs. The approximate weight of one parachute assembly, including the deployment bag, is 513 lbs.

When used as a member of a six-parachute cluster, the parachute is capable of opening without structural failure when released at a speed of 223 ft/sec and a dynamic pressure of $76.3 \, \mathrm{lb/ft^2}$.

d. Margins of Safety

The margins of safety for structural loads in the above-cited opening environment are positive for all components.

3. GORE LAYOUT

The material used for the canopy of a parachute assembly is usually taken from a roll, which is of a given number of running yards (in length) and of a given number of inches in

width. For a solid-flat circular canopy, the number of gores required normally dictates the width of the fabric. However, for the parachute configuration under design, other considerations dominated, primarily the decision to use Military-Specification materials for all fabric components. To adhere to this requirement for suspension lines, it became necessary to assign to the assembly 160 gores. Therefore, the selected prototype parachute assembly is a 135-ft-nominal-diameter solid-flat circular configuration having 160 gores.

a. Basic Gore Geometry

The geometry associated with a basic gore for a solid-flat circular canopy is depicted in Fig. 2. This geometry is representative of the gore's desired finished dimensions.

The theoretical length of the gore is given by

$$R = D_0/2.$$
 (3-1)

The vent radius can be expressed as a fraction of the theoretical gore length. Hence,

$$V_{R} = aR. (3-2)$$

The angle subtended by the intersection of the two theoretical lengths of the gore at the parachute's theoretical center is simply defined by

$$2\phi = \frac{360}{2N}$$
, (3-3)

where N is the number of gores comprising the particular configuration.

The widths of the gore at the vent and skirt respectively are given by

$$M = 2V_{R} \sin \phi \qquad (3-4)$$

and

$$\overline{O} = 2R \sin \phi. \tag{3-5}$$

The actual length and height of the gore are given, respectively by

$$G = R - V_R \tag{3-6}$$

and

$$T = G \cos \phi. \tag{3-7}$$

Table 1 lists the geometry associated with the basic gore for the prototype parachute assembly under design herein.

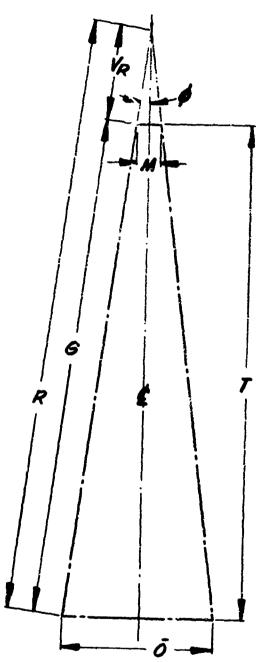


Fig. 2. Geometry associated with a basic gore of a solid-flat circular canopy.

TABLE 1 GEOMETRY ASSOCIATED WITH BASIC GORE OF A 135-ft-D_O

SOLID-FLAT CIRCULAR CANOPY OF 160 GORES

a = 0.04 N = 160 gores D_O = 135 ft R = 67.5 ft V_R = 32.4 in. \$\phi\$ = 1°7'30" M = 1.272 in. \$\overline{\overline{O}}\$ = 31.801 in. \$\overline{G}\$ = 64.800 ft \$\overline{T}\$ = 64.788 ft

b. Fullness and Seam Allowances

The geometry associated with a basic gore, including fullness allowance and seam allowance, for a solid-flat circular canopy is depicted in Fig. 3.

(1) Fullness

The dimensions for the gore widths at the vent and skirt respectively become

$$M_{f} = M(1 + f_{M}) \tag{3-8}$$

and

$$\overline{O}_{\mathbf{f}} = \overline{O} (1 + f_{\mathbf{m}}), \qquad (3-9)$$

where the terms \mathbb{S}_m and $\mathbf{f}_{\overline{O}}$ represent respective fullness factors.

The dimensions for the gore's actual length and height become respectively

$$G_{f} = G (1 + f_{G})$$
 (3-10)

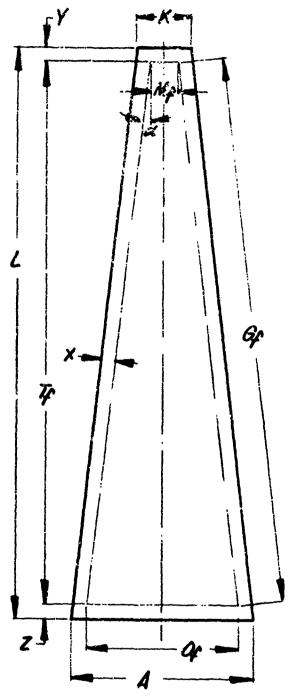


Fig. 3. Geometry associated with a basic gore, including fullness allowance and seam allowance, for a sclid-flat circular canopy.

TABLE 2
GEOMETRY
ASSOCIATED WITH
BASIC GORE,
INCLUDING FULLNESS ALLOWANCE,
FOR A 135-ft-Do

SOLID-FLAT CIRCULAR CANOPY OF 160 GORES

$$f_m = 0.10$$
 $f_{\overline{0}} = 0.0$
 $f_G = 0.0$
 $M_f = 1.399 \text{ in.}$
 $\overline{O}_f = 31.801 \text{ in.}$
 $G_f = 64.800 \text{ ft}$
 $T_f = 64.788 \text{ ft}$
 $\alpha = 1^{\circ}7^{\circ}0^{\circ}$

and

$$T_{f} = G_{f} \cos \alpha, \qquad (3-11)$$

where the term f_{G} represents a fullness factor and

$$\alpha = \sin^{-1} \frac{\overline{O}_{f} - M_{f}}{2G_{f}}.$$
 (3-12)

Table 2 lists the geometry associated with the basic gore, including the fullness allowance, for the prototype parachute assembly under design herein.

(2) Seams

The dimensions for the gore's widths at the vent and skirt become respectively

$$K = M_f + 2\left(\frac{X}{\cos \alpha} + Y \tan \alpha\right)$$
 (3-13)

$$A = \overline{O}_{f} + 2\left(\frac{X}{\cos \alpha} + Z \tan \alpha\right), \qquad (3-14)$$

TABLE 3
GEOMETRY
ASSOCIATED WITH
BASIC GORE,
INCLUDING FULLNESS ALLOWANCE
AND SEAM
ALLOWANCE, FOR
A 135-ft-Do

SOLID-FLAT CIRCULAR CANOPY OF 160 GORES

X = 1.5 in. Y = 2.0 in. Z = 2.0 in. K = 4.322 in. A = 34.879 in. L = 65.121 ft

where the term X represents the seam allowance along the gore's length and Y and Z represent the seam allowance at the gore's two widths.

The dimension for the gore's height becomes

$$L = T_f + Y + Z.$$
 (3-15)

Table 3 lists the geometry associated with the basic gore, including the fullness allowance and the seam allowance, for the prototype parachute assembly under design.

c. Panels Within the Gore

Owing to the size of a gore, it sometimes becomes impractical to design it as a solid piece of cloth. As a result, the gore must be designed so as to be comprised of panels. Figure 4 depicts the geometry associated with a gore comprised of n panels for a solid-flat circular canopy fabricated from a roll of cloth 36.5 inch wide.

The number of panels comprising the gore can be determined by the expression

$$n = \frac{L - S_1}{S} + 2; \qquad (3-16)$$

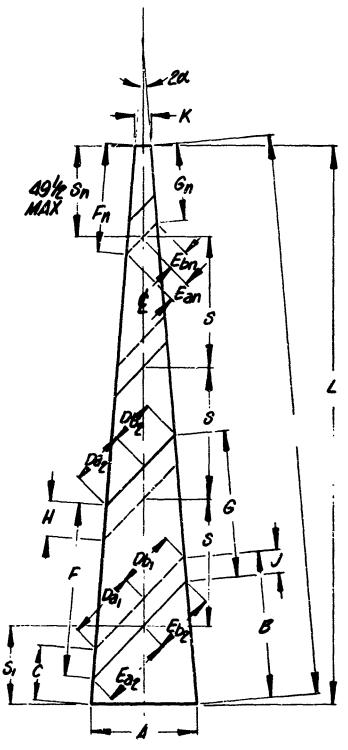


Fig. 4. Geometry associated with an n-panel gore of solid-flat circular canopy, fabricated from a 36.5-in.-wide roll of cloth.

for a 36.5-in.-width fabric and a 1.5-in. seam allowance (X is 1.5 in.),

$$S = 49.498 \text{ in.}$$
 (3-17)

and

$$S_1 = U_1 - 2.121 \text{ in.},$$
 (3-18)

where

$$U_1 = 51.618 \text{ in.} - \frac{1}{2}A.$$
 (3-19)

(1) The First Panel

The geometry associated with the first panel of a gore comprised of n panels for a solid-flat circ lar canopy fabricated from a 36.5-in.-wide roll of cloth is __picted in Fig. 5. Referring to this figure, it can be seen that

$$\beta = 45^{\circ} + \alpha, \qquad (3-20)$$

$$\gamma = 45^{\circ} - \alpha,$$
 (3-21)

$$U_1 = 51.618 - \frac{1}{2}A,$$
 (3-19)

$$S_1 = U_1 - 2.121,$$
 (3-18)

$$B = \frac{36.5}{\sin 6}, \tag{3-22}$$

and, finally,

$$c = \frac{36.5 - 0.707A}{\sin \gamma}.$$
 (3-23)

Now it is possible to present the expressions for the first panel's geometry, that is

$$D_{a_1} = 0.707A - \frac{C \sin \alpha}{0.707}$$
 (3-24)

and

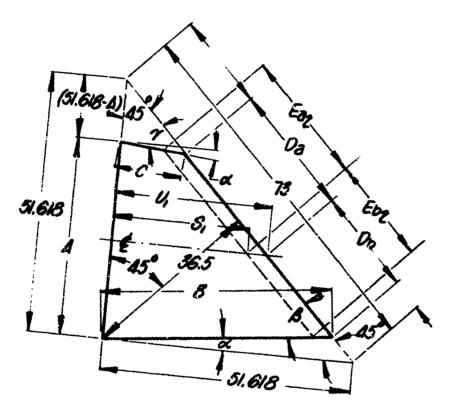
$$D_{b_1} = 0.707A - \frac{51.618 \sin \alpha}{\sin \beta}. \qquad (3-25)$$

The geometry associated with the first panel of the gore for the canopy under design is listed in Table 4(b).

(2) The Second Panel Through the (n - 1) st Panel

The geometry associated with the second panel through the (n-1)st panel of a gore comprised of n panels for a solid-flat circular canopy fabricated from a 36.5-in.—wide roll of cloth is depicted in Fig. 6. Referring to this figure, it can be seen that

$$W = 36.5 \text{ in.},$$
 (3-26)



Pig. 5. Geometry associated with the first panel of an n-panel gore of a solid-flat circular canopy, rabricated from a 36.5-in.-wide roll of cloth.

TABLE 4

GEOMETRY ASSOCIATED WITH PANELS CUMFRISING A SINGLE GORE OF A 160-GORE, 135-ft-D , SOLID-FLAT CANOPY FABRICATED FROM 36.5-18, CLOTH USING A 1.5-1n, SEAM ALLOWANCE

(a) Defined Parameters

	17.076 in.	36.5 in.	52.655 in.	50.642 in.	2.164 in.	2.081 in.	51.618 in.
) (×		#		*	Ħ
	၁	.≆	ጮ	Ö	=	b	>
	17 panels	49.498 1n.	32.057 in.	34.178 in.	460710"	43°53'0"	50.642 in.
	R	Ħ	Ħ				
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Panel	ं इं	.a	a a	á ^a	ſz,	9	અ
ret	į	ŧ	24,191	23,263	•		32.057
٥.	24.251	23,321	22,796	21,921	9.278	4.681	49.498
'n	22,855	21,979	21,400	20.579	9.278	4,681	49.498
ব	21,460	20.636	20,004	19,236	9.278	4,681	49.498
īU	20.064	10.294	18,608	17.894	9.578	4,681	49,498
છ	18.060	17,951	17.212	16,551	9.278	4.681	49.498
7	17.272	16,609	15,816	15,209	9.278	4,683	49.498
ဆ	15.876	15,266	14.421	13.866	9.278	4,681	49.498
on.	14.430	13.924	13.025	12,524	9.278	4,681	49.498
10	13.085	12,581	11,629	11,181	9.278	4.681	864.64
11	11.689	11.239	10,233	9.839	9.278	4.681	49.498
12	10.293	9.896	8.837	8,496	9.278	4,681	49.498
13	8.897	8.554	7.441	7.154	9.278	4,681	49.498
41	7.501	7.211	940.9	5,811	9.278	4,681	86#°6#
15	6.105	5.869	4.650	4.469	0.278	4.683	49.498
					(cont1	(continued on next	page)

THE THE SECTION OF STREET STRE

(b) Panel Geometry* (continued)

ଯ	854.64	5,932
ల	4.681	4.691
ĺΞ	9.278	9.278
ດີ	3.127	i
a a	3.254	3
ia ia	4.527	3.164
& નો	4.720	3.314
Panel	16	17

*Entries are in inches.

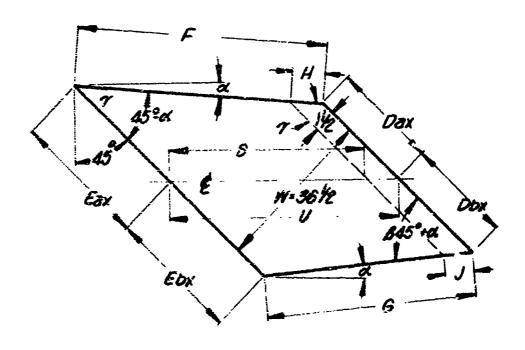


Fig. 6. Geometry associated with the 1st through the (n-1)st panels of an n-panel gore of a solid-flat circular cancey, fabricated from a 36.5-in.-wide roll of cloth.

$$F = \frac{W}{\sin \gamma}, \qquad (3-27)$$

$$G = \frac{W}{\sin \beta}, \qquad (3-28)$$

$$H = \frac{1.5}{\sin \gamma} , \qquad (3-29)$$

$$J = \frac{1.5}{\sin \beta} , \qquad (3-30)$$

$$S = 49.498 \text{ in.},$$
 (3-17)

and

$$U = 51.618 \text{ in.}$$
 (3-31)

The expressions for the second panel's geometry are as follows.

$$E_{a_2} = D_{a_1} + \frac{2.121 \sin \alpha}{\sin \gamma}$$
, (3-32)

$$E_{b_2} = D_{b_1} = \frac{2.121 \sin \alpha}{\sin \beta}$$
, (3-33)

$$D_{a_2} = D_{a_1} = \frac{49.498 \sin \alpha}{\sin \gamma},$$
 (3-34)

and

$$D_{b_2} = D_{b_1} - \frac{49.498 \sin \alpha}{\sin \beta}.$$
 (3-35)

Finally, the general expressions for the geometry of the remaining panels (not including the nth panel) are

$$E_{a_x} = E_{a_2} - (x - 2) \frac{49.498 \sin \alpha}{\sin \gamma},$$
 (3-36)

$$E_{b_x} = E_{b_2} - (x - 2) \frac{49.498 \sin \alpha}{\sin \beta},$$
 (3-37)

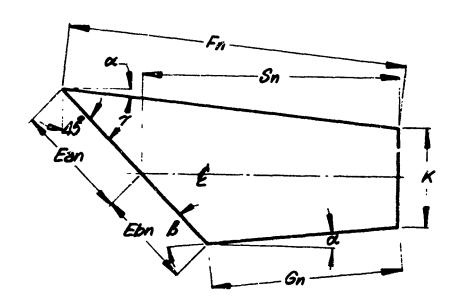
$$D_{a_x} = D_{a_1} - (x - 1) \frac{49.498 \sin \alpha}{\sin \gamma},$$
 (3-38)

and

$$D_{b_x} = D_{b_1} - (x - 1) \frac{49.498 \sin \alpha}{\sin \beta}$$
 (3-39)

It should be cited that the subscript x in the above equations refers to the panel number. The limits of x in these equations are given by

$$2 \le x \le n - 1.$$
 (3-40)



Pig. 7. Geometry associated with the nth panel of an n-panel gore of a solid-flat circular canopy, fabricated from a 36.5-in.-wide roll of cloth.

The geometry associated with the first panel through the (n-1)st panel of the gore for the canopy under design is listed in Table 4(b).

(3) The nth Panel

The geometry associated with the nth panel of a gore comprised of n panels for a solid-flat circular canopy fabricated from a 36.5-in.-wide roll of cloth is depicted in Fig. 7. Referring to this figure, it can be seen that the panel geometry is calculated from the following expressions.

$$E_{a_n} = E_{a_2} - (n-2) \frac{49.498 \sin \alpha}{\sin \gamma},$$
 (3-41)

$$E_{b_n} = E_{b_2} - (n-2) \frac{49.498 \sin \alpha}{\sin \beta}$$
 (3-42)

$$F_{n} = \frac{0.707 (S_{n} + \frac{1}{2}K)}{\sin \gamma}, \qquad (3-43)$$

$$G_n = \frac{0.707 (S_n - \frac{1}{2}K)}{\sin \beta},$$
 (3-44)

and

$$S_n = L - S_1 + S (n - 2).$$
 (3-45)

The geometry associated with the nth panel of the gore for the canopy under design herein is listed in Table 4(b).

4. STRESS ANALYSIS

The purpose of the stress analysis is to establish the margins of safety for the materials selected for use in the prototype parachute assembly. These margins are calculated for the worst-case loading environment; for this particular assembly, such a condition occurs when the assembly is used as a member of a six-parachute cluster to airdrop a 50,000-lb unit load from an aircraft traveling at 223 ft/sec and under a 76.3-lb/ft² dynamic pressure. The maximum reefed opening force experienced by an individual assembly for such an operational environment is 28,300 lb; the method for arriving at this maximum is presented in Volume I, Section 10.

The procedure followed for calculating the margins of safety is, first, to calculate the components' allowable load by use of the expression

allow load =
$$\frac{\text{ult strength}}{\text{overall design factor}}$$
. (4-1)

The ultimate strength is taken from the average of five control samples tested to their ultimate. This is presented in Appendix A. The overall design factor is arrived at by the considerations accounted for in Table 5.

Now, the margin of safety becomes simply

M.S. =
$$\frac{\text{allow load}}{\text{worst-case load}} - 1$$
. (4-2)

The margins of safety calculated for the components of the selected prototype parachute assembly are summarized in Fig. 8.

a. Maximum Canopy Stress for a Vent-pulldown Parachute

With regard to the parachute canopy, it must be pointed out that the establishment of a stress theory is extremely difficult owing to the very nature of the structure. It is n flexible device, constructed from a fabric, and operates in a highly dynamic mode. Therefore, as far as stress analysis is concerned, it is not necessary to attempt to conduct a high-order analytical study. Rather, some basic assumptions were used that, when coupled with experience and intuition, lead to "ball-park" results.

Use of the vent pulldown leads to opening-shape characteristics that somewhat deviate from those normally associated with the standard parachute. This is indicated by a study of movie film depicting deployments of single and clustered G-llA vent pulldowns from an above-terrain altitude of 1500 ft and a release velocity of 150 knots. The general opening shape for all the canopies in these drops is depicted in Fig. 9. This shape is most definitive at or just following full reef, the point at which the parachute loads are at maximum.

Figure 9 shows that, at full reef, the canopy exhibits prominent domes ("false vents"). The true vent is, of course, pulled down within the skirt area. Therefore, there is no physical means for the canopy to bleed off pressure. This accounts for the relatively quick opening and resulting high loads associated with the vent-pulldown parachute.

(1) G-11A Cargo Parachute

The G-llA cargo parachute under discussion here has a reefing ratio of 20%; that is, $D_{\rm R}=0.2D$. This then implies that, at reefed state, the parachute diameter is 20 ft. Figure 10 shows the results of scaling from the frames of the previously mentioned movie film. As can be observed, the scaling was reasonably accurate. Therefore, from this

TABLE 5
VARIOUS CONSIDERATIONS FOR ARRIVING AT OVERALL DESIGN FACTOR

Ites						Design		
•••	Safety, 1	Line conver- gence, ²	Asym- metrical load, 1	Joint eff.,	Abru- sion, b	Pa- tigue, ²	factor, cf/mlk	design factor, jcf/mlk
Canopy	1			 	!			
Cloth	2.00	N/A	1.05	1.00	6.96	0.95	1.15	2.30
Hein seam	2.00	N/A	1.05	0.98	0.96	0.95	1.15	2.36
Cross Hest	2.00	H/A	1.05	0.79	9.96	0.95	1.46	2.92
Suspension line								
To con. link	2.00	1.04	1.05	9.97	0 96	9.95	1.24	2.48
To skirt	2.00	1.04	1.25	1.00	0.40	0.95	1.20	2.40
To main seem at skirt and went	2.00	N/A	1.05	1.00	o.96	0.95	1.15	2.30
To went ring	2.60	N/A	1.05	0.87	0.96	0.95	1.33	2.66
Reefing line	2.00	N/A	1.05	c.96	1.00	1.00	1.09	2.18
Skirt reinf.	2.00	R/A	1.05	0.99	0.96	0.95	1.17	2.34
Vent reinf.	2.00	N/A	1.05	0.93	0.96	0.95	1.24	2.48
Riser				!				
To conn. link	2.00	N/A	1.05	0.75	0.96	0.95	1.53	3.06
To clevia	2.00	37A	1.05	0.88	0.46	0.95	1.31	2.62
ser est.								
7c clevis	2.00	N/A	1.05	0.82	0.95	C.95	1.41	2.82
To load				j	j			
attach.	2.00	N/A	1.05	u.c7	3.40	0.95	1.34	2.68
Splice	2.00	H/A	1.05	58.0	0.96	0.95	1.31	2.62
Center line								
To went ring	2.00	N/A	1.05	0.83	0.96	0.95	1.39	2.77
To clevia	2.90	N/A	1.05	0.87	0.95	0.95	1.32	2.64
Spilce	2.00	N/A	1.05	0.86	0.96	0.95	1.34	2.68

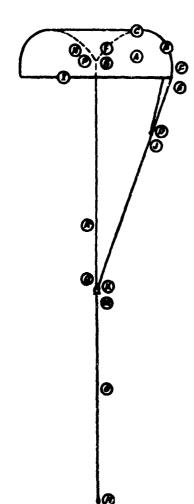
iVol. I, Sec. 3.0.

Ref. 3, p. 370.

Conventional practice.

App. A; for riser extension, see Fig. 4-14.

Selected on basis of having service life similar to that of the T-10.



Symbol	Item	Margin of safety
	Canopy	
A	Cloth	+1.05
В	Main seam	+0.96
С	Cross seam	0
	Suspension line	
D	To conn. link	+0.35
E	To skirt	+0.39
F	In main seam	+0.38
G	To vent ring	+0.19
Н	Vent reinforcing	+0.75
I	Skirt reinforcing	+0.43
	Riser	
J	To conn. link	+0.27
K	To clevis	+0.48
	Riser extension	
M	To clevis	+0.17
N	To load attach.	+0.23
0	Splice	0
	Center line	
P	To vent ring	O
Q	To clevis	+0.04
R	Splice	+0.03

Fig. 8. Summary of the margins of safety.

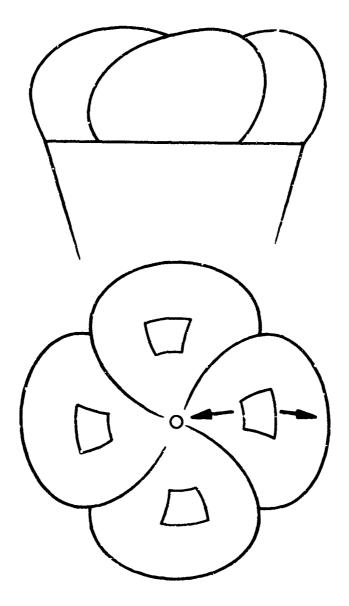


Fig. 9. General shape characteristics associated with the opening of the G-llA vent-pulldown parachute.

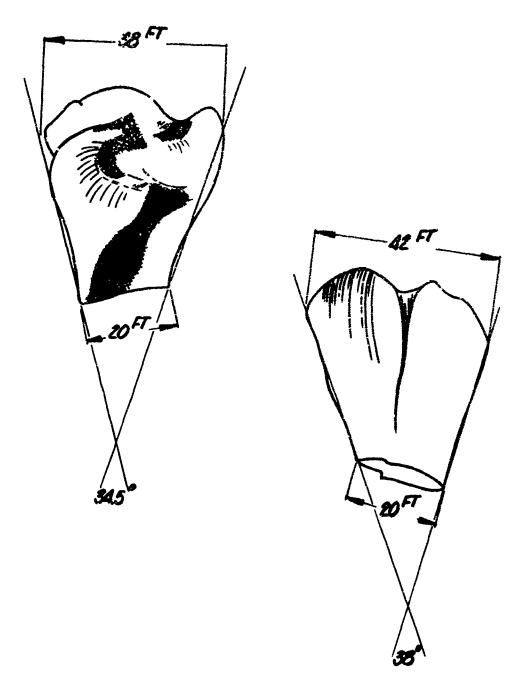
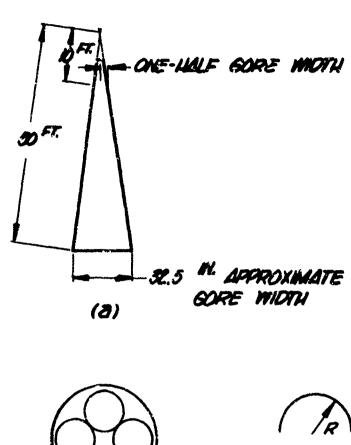


Fig. 10. Dimensions scaled from movie film depicting opening of the G-11A vent-pulldcwn parachute.



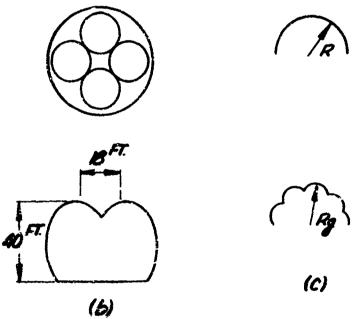


Fig. 11. Approximate dimensions associated with the opening shape of a G-11A vent-pulldown parachate.

figure, it can be established that the high-pressure area of the canopy is located approximately 40 ft from the skirt of the canopy. Figure 11 (a) shows that the width of any individual gore at this location is simply

$$2 \times \frac{10}{50} \times \frac{32.5}{2} = 6.5 \text{ in.}$$
 (4-3)

Since the canopy has 120 gores, this means that, at the location of the high-pressure areas, the circumference can be calculated to be

$$120 \times 6.5 = 7c$$
 in. = 65 ft. (4-4)

From scaling the film, it was determined that the false vents lie on a circumference of a circle whose diameter is approximately 18 ft [see Fig. 11 (b)]. Hence, the circumference is

$$18\pi \approx 56 \text{ ft.}$$
 (4-5)

The difference between the above two circumferences is 9 ft. This means that, at the reefed state, there is some 9 it of fullness, or, for the case of four "false vents," 2.25 ft per false vent. This amounts to some four gores per high-pressure area that have not yet unfolded.

The stress in the canopy is now determined by assuming that each of the high-pressure areas lies on the dome of an 18-ft-diam. hemisphere. Viewed this way, the maximum stress becomes simply the hoop stress; hence,

$$qR = 30 \times 9 = 270 \text{ lb/ft} = 22.4 \text{ lb/in.}, (4-6)$$

where q is the aerodynamic pressure, which El Centro droptest data reveal to be approximately 30 lb/ft^2 at attainment of full reef.

It can be concluded that the above approach to the maximum canopy stress present in the deployment of a vent-pulldown parachute is conservative because, in practice, the main seams carry a significant portion of the parachute load and consequently cut into the smooth hemisphere. From Fig. 11 (c), it becomes obvious that, since R < R, the product of q and R is reduced.

(2) Selected Prototype Parachute Assembly (D = 135 ft)

For a cluster of six 135-ft-diam. vent-pulldown parachutes, it must be assumed that the high-pressure areas each lie on the dome of a hemisphere whose diameter is

$$0.18 \times 135 = 24.3 \text{ ft}$$
 (4-7)

The aerodynamic pressure at the time of maximum cluster load (at or following attainment of full reef) is approximately 28.6 lb/ft² (refer to Appendix B). Hence the maximum canopy stress, which is a circumferential stress acting in a direction normal to the main seam, is

$$aR = 28.6 \times 12.15 = 347 \text{ lb/ft} = 29 \text{ lb/in.} (4-8)$$

Knowing the maximum canopy stress, it is now possible to calculate its margin of safety.

(1 Canopy Cloth

The allowable load on the canopy cloth itself is calculated from Eq. (4-1):

allow load =
$$\frac{136.6 \text{ lb/in.}}{2.30}$$
 = 59.4 lb/in. (4-1a)

Use of Eq. (4-2) yields a margin of safety of

M.S. =
$$\frac{59.4 \text{ lb/in.}}{29 \text{ lb/in.}} - 1 = +1.05$$
, (4-2a)

where (it should be noted), for the canopy, the corst-case load is the maximum canopy stress.

(2 Canopy Main Seam

The allowable load on the canopy main seam is calculated from use of Eq. (4-1). Hence,

allow load =
$$\frac{134 \text{ lb/in}}{2.36}$$
 = 56.8 lb/in. (4-1b)

Use of Eq. (4-2) yields, for the margin of

safety

M.S. =
$$\frac{56.8 \text{ lb/in.}}{29 \text{ lb/in.}} - 1 = +0.96.$$
 (4-2b)

(3 Canopy Cross Seam

Once more, Eq. (4-1) yields the allowable load, this time for the canopy cross seam:

allow load =
$$\frac{60 \text{ lb/in}}{2.92}$$
 = 20.5 lb/in. (4-1c)

Since the cross seam is located on the gore so as to subtend an angle of 45° with the circumerential refer-

ence, and since the canopy stresses in the direction normal to the circumference are assumed negligible, it is permissible to state that the force normal to the cross seam is simply

(max canopy stress)
$$\times$$
 sin 45°
= (29 lb/in.) \times 0.707 = 20.5 lb/in. (4-9)

Equation (4-2) leads to the following calculated margin of safety:

M.S. =
$$\frac{20.5 \text{ lb/in.}}{20.5 \text{ lb/in.}} - 1 = 0.$$
 (4-2c)

b. Reefing Line

The forces to which the reefing line is subjected are a function of a number of fixed factors: specifically, the number of gores in a parachute, the maximum opening load, and the length of the suspension lines. These forces in turn are functions of certain variables which may not be determined without a drop-test program—such as the most suitable reefing ratio and the inflated shape of the reefed parachute as typified by the angle formed between the tangent to the radial seam at the skirt and a reference parallel to the parachute center line.

(1) Recfing-line Force

Figure 12 depicts the force behavior of the reefing line for a vent-pulldown parachute at the initial stages of full reefed condition. If $F_{\rm O}$ represents the maximum load, then the force in one suspension line becomes

$$(F_{SL})_{FR} = \frac{F_O}{N \cos \alpha_{FR}}, \qquad (4-10)$$

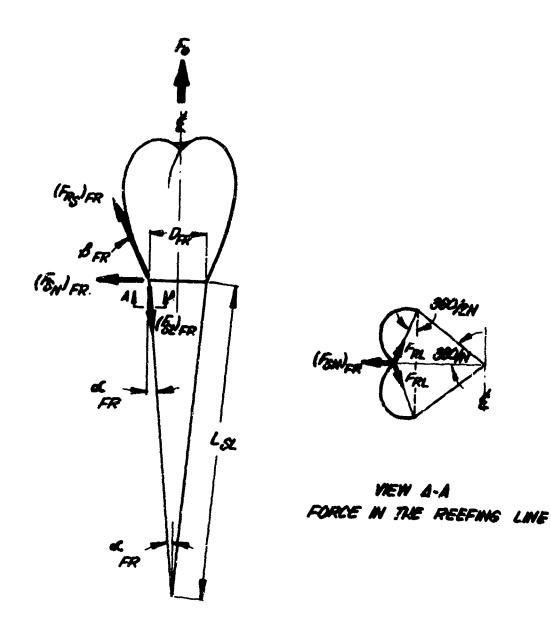
where N is the number of gores comprising the parachute canopy and is also equal to the number of suspension lines.

The component of the suspension-line force in a direction parallel to the parachute center line is simply

$$\left(F_{SL_{v}}\right)_{FR} = \frac{F_{O}}{N}, \qquad (4-11)$$

and the component normal to the center line is

$$(F_{SL_H})_{FR} = \frac{F_0}{N} \tan \alpha_{FR}.$$
 (4-12)



PARACUUTE FORCES

Fig. 12. Assumed force behavior of full reef for the reefing line of a parachute employing the vent-pulldown technique.

For the parachute to be in reefed equilibrium,

$$(F_{R_V})_S = (F_{SL_V})_{FR}.$$
 (4-13)

The force in the main seam radially directed from the skirt is

$$(F_R)_S = \frac{F_O}{N \cos \beta_{FR}}. \qquad (4-14)$$

It can be shown that

$$(F_{R_H})_S = \frac{F_O}{N}$$
 tan β_{FR} . (4-15)

The force that normally tends to open the mouth of the parachute is $(F_S)_{FR}$, and can be expressed as

$$(F_{S_N})_{FR} = (F_{R_H})_{S} - (F_{SL_H})_{FO}.$$
 (4-16)

Substituting Eqs. (4-12) and (4-15) into Eq. (4-16) yields

$$(F_{S_N})_{FR} = \frac{F_O}{N} \quad (\tan \beta - \tan \alpha)_{FR}.$$
 (4-17)

If it is desired to use reefing, it can be seen that the force acting directly on the reefing rings (as a result of the reefing line's resistance to the opening tendencies) has a magnitude equal to $(F_S)_{FR}$ but opposite in direction.

The force in the reefing line can now be simply expressed as

$$F_{RL} = \frac{(F_{S_N})_{FR}}{2 \sin (360/2N)}$$
 (4-18)

Substituting Eq. (4-17) into Eq. (4-18) yields

$$\frac{F_{RL}}{F_C} = \frac{(\tan \beta - \tan \alpha)_{FR}}{2N \sin (360/2)}$$
 (4-19)

However, it becomes desirable to express the forces in the reefing line in terms of the reefing ratio. From Fig. 12, it can be seen that

$$\alpha = \sin^{-1} \frac{L_{D_R}}{L_{SL}}. \qquad (4-20)$$

The reefing ratio is defined by

$$R_{FR} = \frac{D_{FR}}{D_O} , \qquad (4-21)$$

where $D_{\rm R}$ is the steady-state diameter, and $D_{\rm O}$ is the diameter of the flattened canopy. Now, substituting the latest two equations into Eq. (4-19) finally leads to

$$\frac{F_{RL}}{F_{O}} = \frac{\tan \beta_{FR} - \tan [\sin^{-1} (RD_{O}/2L_{SL})]_{FR}}{2N \sin (360/2N)}.(4-22)$$

(2) Selected Prototype Parachute Assembly

Figure 13 depicts the curves plotting Eq. (4-22) for various β -values. These curves are based upon the geometry associated with the selected prototype parachute assembly; that is,

$$D_0 = 135 \text{ ft},$$

$$L_{SL} \simeq 169 \text{ ft,}$$

and

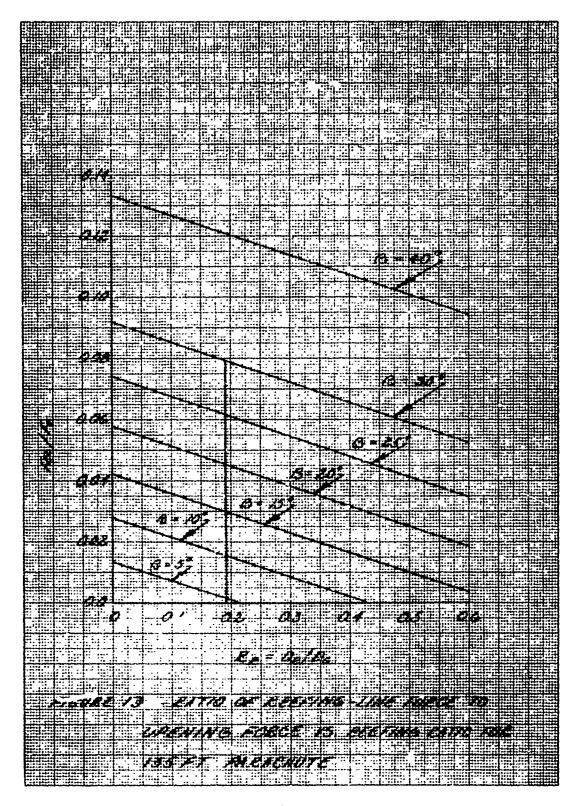
$$N = 160$$
 gores.

Referring to Fig. 10, it can be seen that $\beta_{PR} = 18^{\circ}$. In addition, the established reefing ratio, R_{PR} , for the selected prototype is approximately 19.1% (Volume I, Section 10.2, p. 161). Hence, from Fig. 13,

$$\frac{F_{RL}}{F_O} = 0.042$$
. (4-23)

Since, for this case, F_C is 28,300 lb, then

$$F_{RL} = 1187 \text{ lb.}$$
 (4-23a)



Now it is possible to arrive at the allowable load on the reefing line. From Eq. (4-1),

allow load =
$$\frac{2749 \text{ lb}}{2.18}$$
 = 1261 lb. (4-ld)

From Eq. (4-2), the margin of safety is calculated to be

M.S. =
$$\frac{1261 \text{ lb}}{1187 \text{ lb}} - 1 = +0.06$$
. (4-2d)

c. Skirt-reinforcing Band

It can be reasonably assumed that the skirt-rein-forcing band experiences its maximum force at approximately full open since, in this condition, the scalloped shape associated with the skirt is minimized. Reference to Volume I, p. 164, reveals that it is also reasonable to assume that the maximum parachute force is approximately the same for full open as for full reef.

(1) Load Experienced by the Skirt-reinforcing Band

Figure 14 depicts the assumed force behavior at full open for the skirt of a parachute employing the vent-pulldown technique. Using rationale similar to that exemplified in calculating the reefing-line force enables the calculation of the maximum load experienced by the skirt-reinforcing cand. To do this, the very conservative assumption is made that there is no scalloping effect at the skirt.

A portion of the aerodynamic force acting on the gore is resisted at the skirt. This resistance is equal to the horizontal component of the suspension-line force,

$$(F_{SL_u})_{FO} = \frac{(F_{SL_v})_{FO}}{N} \tan \alpha_{FO},$$
 (4-24)

View A-A of Fig. 14 shows that the force in the skirt-reinforcing band at full open becomes

$$(F_{SB})_{FO} = \frac{(F_{SL_H})_{FO}}{2 \sin (360/2N)}$$
 (4-25)

Substituting Eq. (4-24) into Eq. (4-25) yields

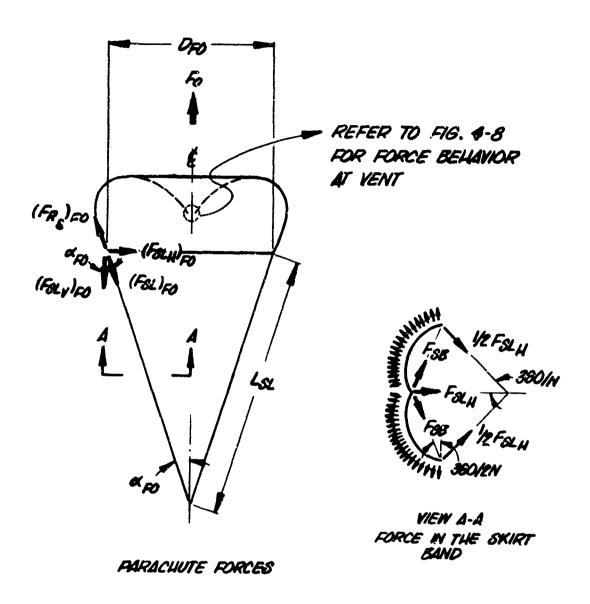


Fig. 14. Assumed force behavior at full open for the skirt of a parachute employing the vent-pulldown technique.

$$(F_{SB})_{FO} = \frac{(F_{SL_V})_{FO} \tan \alpha_{FO}}{2 \sin (360/2N)}$$
 (4-25)

(2) Selected Prototype Parachute Assembly

The geometry associated with the prototype parachute assembly, shown at full open in Fig. 14, is as follows.

$$D_{FO} \simeq 96 \text{ ft},$$

$$L_{SL} \simeq 169 \text{ ft},$$

and

N = 160 gores.

From this information, it can be seen that

$$\alpha_{FO} = \sin^{-1} \frac{48 \text{ ft}}{169 \text{ ft}}$$
 (4-27)

or

$$\alpha_{FO} = 16.5^{\circ}.$$
 (4-28)

Use of Eq. (4-26), if the maximum force at full open is 28,300 lb, yields

$$(P_{SB})_{FO} = \frac{28,300 \text{ tan } 16.5^{\circ}}{2 \times 160 \times \sin (360/2N)} = 1375 \text{ ib.}(4-26a)$$

The allowable load on the skirt-reinforcing band is calculated by Eq. (4-1). Hence,

allow load =
$$\frac{4598 \text{ lb}}{2.34}$$
 = 1960 lb. (4-le)

From Eq. (4-2), the margin of safety is calculated to be

M.S. =
$$\frac{1960 \text{ lb}}{1375 \text{ lb}} - 1 = \pm 0.43.$$
 (4-2e)

d. Vent-reinforcing Band

It can be reasonably assumed that the vent-reinforcing band experiences its maximum force at approximately full open, since in this condition the scalloped shape associated with the vent is minimized.

(1) Load Experienced by the Vent-reinforcing Band

Referring to Fig. 1^4 , it can be assumed that at full open the force in the radial main seam at the skirt is equal to the force in the suspension line. Hence,

$$(F_{R_S})_{FO} = (F_{SL})_{FO}.$$
 (4-29)

Carrying this reasoning further, one can state that the force or tension in that portion of the suspension line that traverses the main seam is constant throughout. Therefore, it can be assumed that, at full open, force in the main radial seam at the skirt is equal to the force in the main radial seam at the vent:

$$(F_{R_S})_{FO} = (F_{R_{VL}})_{FO}.$$
 (4-30)

Substituting Eq. (4-29) into Eq. (4-30) yields

$$(F_{R_{vt}})_{FO} = (F_{SL})_{FO}.$$
 (4-31)

The algebraic summation of the horizontal components of $(F_{R_{vt}})_{FO}$ and the vent-line force $(F_{VL})_{FO}$ yields

the value of the normal force pulling outward on the ventreinforcing band, creating its tension load:

$$(F_{vt_N})_{FO} = (F_{R_{vt_H}})_{FO} - (F_{VL_H})_{FO}.$$
 (4-32)

From reasoning similar to that used for the reefing line, the tension or force in the vent-reinforcing band at full open can now be conservatively stated as

$$(F_{VB})_{FO} = \frac{(F_{Vt_N})_{FO}}{2 \sin (360/2N)}$$
 (4-33)

Substituting Eq. (4-32) into Eq. (4-33) yields

$$(F_{VB})_{FO} = \frac{(F_{Vt_H})_{FO} - (F_{VL_H})_{FC}}{2 \sin (360/2N)}$$
 (4-33a)

From Fig. 15, it can be seen that

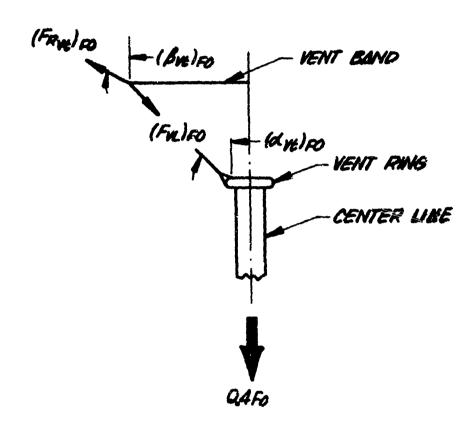


Fig. 15. Assumed force behavior at full open for the vent of a parachute employing the vent-pulldown technique.

$$(F_{R_{vt_H}})_{FO} = (F_{R_{vt_V}})_{FO} \tan (\beta_{vt})_{FO}$$
 (4-34)

and

$$(F_{VL_{H}})_{FO} = (F_{VL_{V}})_{FO} \tan (\alpha_{vt})_{FO}.$$
 (4-35)

Substitution of Eqs. (4-34) and (4-35) into Eq. (4-33a) yields

$$(F_{VB})_{FO} = \frac{(F_{R_{V^{\pm},V}})_{FO} \tan (\beta_{Vt})_{FO} - (F_{VL_{V}})_{FO} \tan (\alpha_{Vt})_{FO}}{2 \sin (360/2N)}.$$
(4-33b)

To satisfy equilibrium conditions,

$$(F_{R_{vt_{V}}})_{FO} = (F_{VL_{V}})_{FO},$$
 (4-36)

and Eq. (4-33b) becomes

$$(F_{VB})_{FO} = \frac{(F_{VL_V})_{FO} \left[\tan (\beta_{vt})_{FO} - \tan (\alpha_{vt})_{FO} \right]}{2 \sin (360/2N)}$$
 (4-33c)

The vertical component of the vent-line force at full open can be expressed as

$$(F_{VL_V})_{FO} = \frac{0.4 F_O}{N}$$
, (4-37)

and the angle $(\beta_{vt})_{FC}$ at full open as

$$(\beta_{vt})_{FO} = \cos^{-1} \frac{(F_{R_{vt}})_{FO}}{(F_{R_{vt}})_{FO}}.$$
 (4-38)

Substitution of Eqs. (4-31) and [4-36] into Eq. (4-38) yields

$$(\beta_{vt})_{FO} = \cos^{-1} \frac{(F_{VL_V})_{FO}}{(F_{SL})_{FO}}$$
 (4-38a)

From Fig. 14, it can be seen that

$$(F_{SL_V})_{FO} = \frac{(F_{SL})_{FO}}{\cos \alpha_{FO}}.$$
 (4-39)

Since

$$(F_{SL_V})_{FO} = \frac{F_O}{N}, \qquad (4-40)$$

the expression for $(\beta_{vt})_{FC}$ is reduced to

$$(\beta_{Vt})_{FO} = \cos^{-1} \left[\frac{N(F_{VL_V})_{FO}}{F_O} \cos \alpha_{FO} \right].$$
 (4-38b)

Substituting Eqs. (4-37) and (4-38b) into Eq. (4-33) finally yields the force in the vent band at full open:

$$(F_{VB})_{FO} = \frac{F_0 \{ \tan [\cos^{-1} (0.4N \cos \alpha_{FO})] - \tan (\alpha_{vt})_{FO} \}}{5N \sin (360/2N)}$$
 (4-33d)

In the prototype parachute assembly under study, the following information is characteristic at full open:

$$F_C = 28,300 \text{ lb},$$

N = 160 gores

$$\alpha_{FO} = 16.5^{\circ}$$
,

and

$$(\alpha_{vt})_{FO} = 45^{\circ}$$
.

Substitution of these values into Eq. (4-33d)

yields

$$(F_{VB})_{FO} = 1060 \text{ lb.}$$
 (4-41)

The allowable load on the vent-reinforcing band is calculated by Eq. (4-1). This leads to

allow load =
$$\frac{4598 \text{ lb}}{2.48}$$
 = 1850 lb. (4-1f)

Use of Eq. (4-2) leads to the following margin of safety.

M.S. =
$$\frac{1850 \text{ lb}}{1060 \text{ lb}} - 1 = +0.75$$
. (4-2f)

e. Suspension Lines

The ultimate load of the suspension lines is calculated from the expression

ult load = (no. of lines) × (line strength).(4-42)

For the case herein,

no. of lines = 160

and

line strength = 591 lb;

hence,

ult strength =
$$(160 \text{ lines}) \times (591 \text{ lb/line}) = 94,500 \text{ lb.}$$
 $(4-42a)$

(1) Suspension Lines Joined at Connector Links

Equation (4-1) yields the allowable load on the suspension lines based upon the joint at the connector links:

allow load =
$$\frac{94,500 \text{ lb}}{2.48}$$
 = 38,200 lb. (4-lg)

The margin of safety, from Eq. (4-2), becomes

M.S. =
$$\frac{38,200 \text{ lb}}{28,300 \text{ lb}} - 1 = +0.35$$
. (4-2g)

(2) Suspension Lines Joined at Skirt

The allowable load on the suspension lines based upon the joint at the skirt is

allow load =
$$\frac{94,500 \text{ lb}}{2,40}$$
 = 39,400 lb. (4-1h)

Therefore the margin of safety is

M. S. =
$$\frac{39,400 \text{ lb}}{28,300 \text{ lb}} - 1 = +6.39$$
 (4-2h)

(3) Suspension Lines in the Main Seam Joined at the Skirt and Vent

The allowable load at the skirt and vent on the suspension lines in the main seam (since the suspension lines

run through the main seam on up to the arex) is

allow load =
$$\frac{94.500 \text{ jb}}{2.30}$$
 = 41,150 lb. (4-11)

The margin of safety becomes

M.S. =
$$\frac{41,150 \text{ lb}}{29,900 \text{ lb}} - 1 = +0.38$$
, (4-21)

where the total suspension-line worst-case load is calculated from use of a variation of Eq. (4-14):

worst-case load *
$$\frac{P_0}{\cos \beta_{FE}}$$
. (4-43)

For this case,

$$F_0 = 28,300 \text{ lb},$$

and

$$\beta \approx 18^{\circ}$$
.

Hence,

worst-case load =
$$\frac{28,300 \text{ lb}}{\cos 18^{\circ}}$$
 = 29,900 lb. (4-43a)

(4) Suspension Lines Joined at the Vent Ring

Equation (4-1) yields the allowable load on the vent lines (suspension lines running from the vent band to the vent ring):

allow load =
$$\frac{94,500 \text{ lb}}{2.66}$$
 = 35,500 lb. (4-1j)

The extreme worst-case load that could be experienced by the vent lines is 29,900 lb. Hence, the margin of safety for this condition becomes

M.S. =
$$\frac{35,500 \text{ lb}}{29,900 \text{ lb}} - 1 = +0.19$$
. (4-2j)

f. Riser

(1) Riser Joint at the Connector Link

Equation (4-1) yields the allowable load for the riser at the connector link:

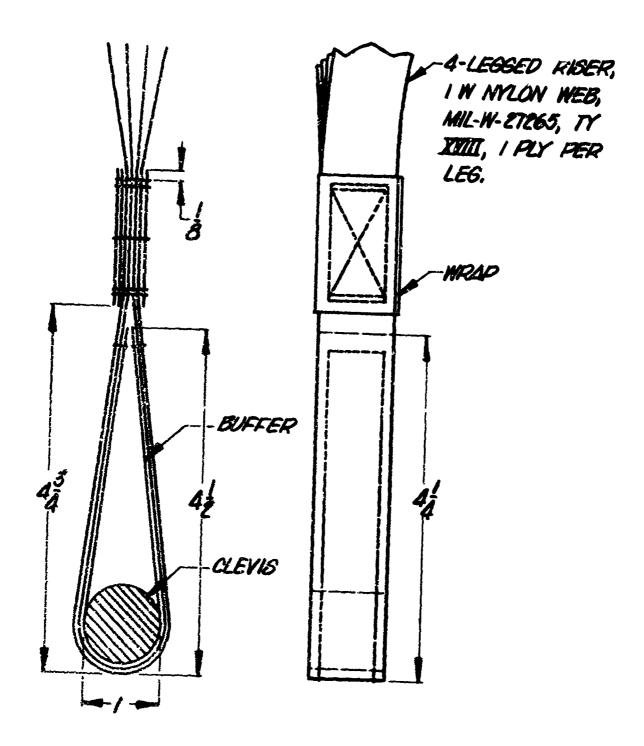


Fig. 16. Riser joint at clevis (refer to Satick Laboratories) Dwg. Xll-1-1651).

allow load =
$$\frac{7253 \text{ lb}}{3.06}$$
 = 2475 lb. (4-lk)

The margin of safety is

M.S. =
$$\frac{2475}{1870 \text{ lb}} - 1 = +0.27$$
, (4-2k)

where the worst-case load experienced by the riser is based on the total worst-case load experienced by the suspension lines. The latter case is stated in Eq. (4-43a). Since ten suspension lines feed into each riser via the connector link, the riser worst-case load becomes

worst-case load =
$$\frac{29,900 \text{ lb}}{160 \text{ lines}}$$
 × (10 lines/riser)
= 1870 lb/riser. (4-44)

(2) Riser Joint at the Clevis

The allowable load for the riser at the clevis is

allow load =
$$\frac{(7253 \text{ lb}) \times (4 \text{ ply})}{2.62}$$
 = 11,100 lb.(4-1m)

The margin of safety becomes simply

M.S. =
$$\frac{11,190 \text{ lb}}{(1870 \text{ lb}) \times (4 \text{ ply})} - 1 = +0.46$$
 (4-2m)

g. Crater Line

(1) Center-line Joint at the Vent Ring

Equation (4-1) yields the allowable load for the center-line at the vent ring:

allow load =
$$\frac{(15,512 \text{ lb}) \times (2 \text{ ply})}{2.77} = 11,200 \text{ lb.}$$
 (4-lp)

From Eq. (4-2), the margin of safety is

M.S. =
$$\frac{11,200 \text{ lb}}{11,300 \text{ lb}} - 1 = 0$$
, (4-2p)

where from Volume 1, p. 176, it can be seen that the centerline werst-case load is given by

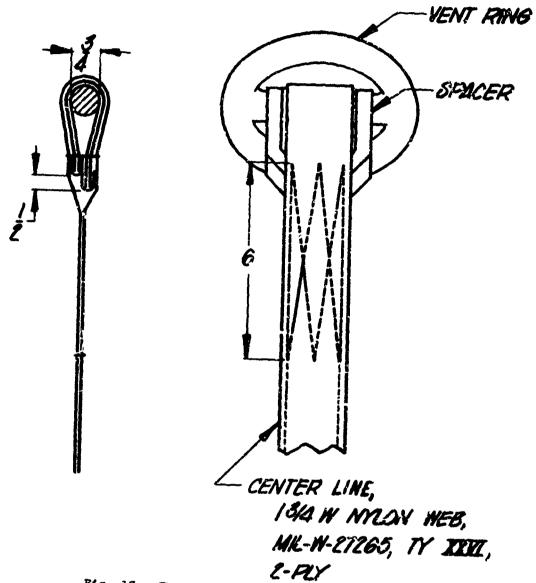


Fig. 17. The center-line joint at the vent ring.

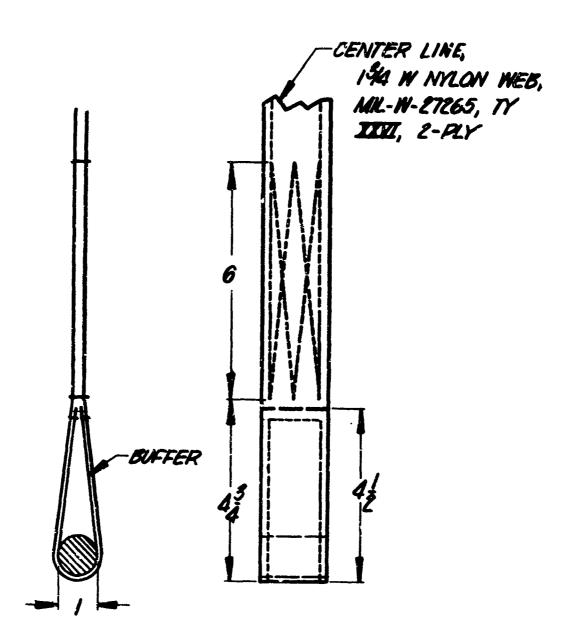


Fig. 18. Center-line joint at clevis.

worst-case load = $0.4 \times (28,300 \text{ lb}) = 11,300 \text{ lb}$. (4-47)

(2) Center-line Joint at the Clevis

The allowable load on the center line at the clevis is

allow load =
$$\frac{(15,512 \text{ lb}) \times (2 \text{ ply})}{2.64}$$
 = 11,750 lb. (4-lq)

The margin of safety is

M.S. =
$$\frac{11,750 \text{ lb}}{11,300 \text{ lb}} - 1 = +0.4$$
. (4-2q)

(3) Center-line Splice

(1 Margin of Safety on the Basis of Theory

Figure 19 reveals that the length of stitching for the splice is 11-5/8 in. If there are approximately 7 stitches sewn per inch, then for 8 rows of stitching there is a total of

$$(11-5/8 \text{ in./row}) \times (7 \text{ stitches/in.}) \times (8 \text{ rows})$$
= 650 stitches.

The rated ultimate tensile strength of the thread is 50 lb. This means that, for 650 stitches, the following ultimate tensile load can 32 developed:

$$(650 \text{ stitches}) \times (50 \text{ lb/stitch}) = 32,550 \text{ lb};$$

however, the efficiency of the thread is approximately 75%, and the abrasion and fatigue factors are 0.96 and 0.95, respectively. Noting that the safety factor is 2.^, then the thread's overall design factor becomes

$$\frac{2.0}{0.75 \times 0.96 \times 0.95} = 2.92.$$

Now, using Eq. (4-1), it becomes possible to arrive at the theoretical allowable load for the splice:

allow load =
$$\frac{32,550 \text{ lb}}{2.92}$$
 = 11,120 lb. (4-45)

The margin of safety for the splice becomes

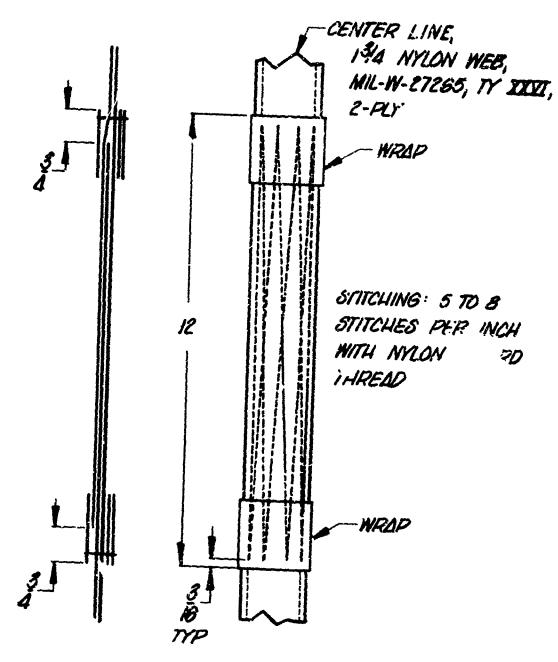


Fig. 19. Splice for the center line and main riser. (Wrap for center line is 1-3/4 W mylor web, Ty XII, for main riser, wrap is 1-3/4 W mylon web, Ty XIII; both MIL-W-27265.)

M.S. =
$$\frac{11,120 \text{ lb}}{11,300 \text{ lb}} - 1 = -0.02$$
. (4-46)

The negative margin of safety based on theoretical determination of the efficiency of the center-line splice indicates that the center-line splice may be marginal. Hence, it becomes necessary to conduct structural tests to determine the actual efficiency of the splice.

(2 Margin of Safety on the Basis of Tests

Appendix A shows that testing reveals the allowable load for the center-line splice to be

allow load =
$$\frac{(15,512 \text{ lb}) \times (2 \text{ ply})}{2.68}$$
 = 11,600 lb. (4-ir)

The margin of safety, therefore, becomes

M.S. =
$$\frac{11,600 \text{ lb}}{11,300 \text{ lb}} - 1 = +0.03.$$
 (4-2r)

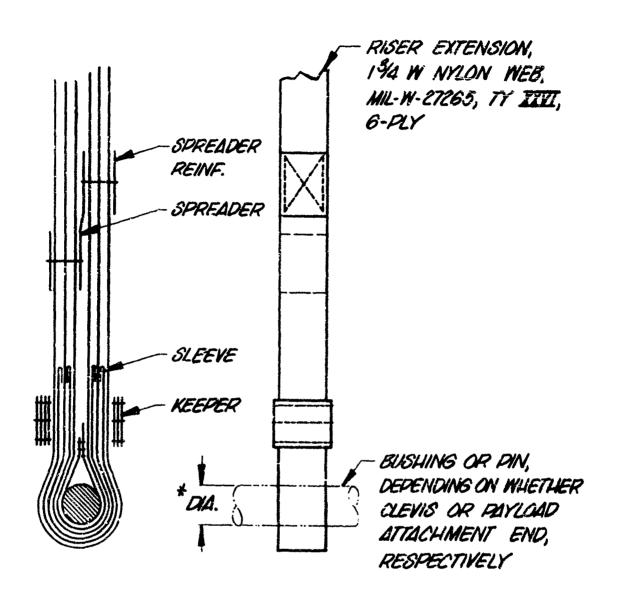
This margin of safety is higher than that arrived at from theory (Eq. 4-46); hence, it can be concluded from the results of Eqs. (4-46) and (4-2r) that the margin of safety derived from theory is reasonably conservative.

h. Riser Extension

The riser extension (or main riser) is of 6-ply webbing. Because of the high anticipated ultimate strength of the riser extension (of the order of 90,000 lb) destruct testing at Pioneer's laboratory facilities becomes infeasible. As a result, the means for arriving at this configuration's joint efficiency is accomplished by theoretical calculations based on those presented in ref. 4 and summarized by the curve depicted in Fig. 21.

Table 6 presents the results of the above theory. It shows that the correlation between the theoretical joint efficiencies and the test joint efficiencies for both the riser and center line compare favorably. In fact, the theory is somewhat conservative. As a result, it can be stated with confidence that the riser-extension joint efficiencies arrived at through the theory presented in ref. 4 are reasonably realistic.

In Fig. 21, the curve defines the joint efficiency of a webbing around a pin. The parameter is given by the expression $\bf r_i$ /Nt, where N is the number of webbing plies, $\bf r_i$



* FOR CLEVIS JOINT, HAVE 2^{IN.}-DIA. BUSHING; FOR PAYLOAD-ATTACHMENT JOINT, HAVE 3^{IN.}-DIA. PIN

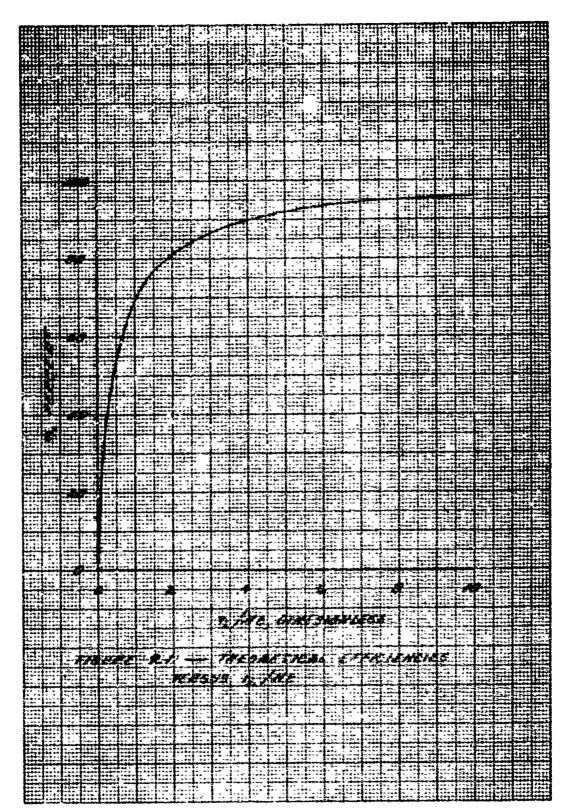
Fig. 20. Fiser-extension joints at clevis and payload-attachment ends.

TABLE 6 COMPARISON OF EFFICIENCIES BASED ON THEORY AND TEST

Item	r, in.		.3	t, in.	•	r, /.it	n, in.	Test eff %
Riser	ı					4		
Conn. 11nk	0.164	H	4	0.094		1.75	75	75
Clevis	0.50	tu	23	0.094		2,66	85	33
Conter line								
Vent ring	0.37	-		0.156		2.40	83.5	83.4
Clevis	1.0	rt		0.156		6.40	*†6	76
Riser extension								
Clevis bushing	1,00	i.J		0.156		2.14	82	i
Payload attachment	1,50	n		0.156		3,21	82	9
*Although the actual cle	clevis	radius	13 0	.50 1n.	the	structual	test of the	evis radius is 0.50 in. the structual test of the ettachment of

50

As shown, the comparison between the efficiency predicted by theory (n) and that arrived at by test is excellent. Henco, the efficiency of the attachment of the center line to the clevis by a pin of 0.50-in. radius can be replistically established as 87%, based on the curve depicted in Fig. 11. the center line to the clevis was conducted with a pin of 1.00-in. radius (ri = 1.00).



is the inside radius of the innermost ply of webbing (in inches), and t is the webbing's thickness (in inches). The curve fails to account for the reinforcing effect of any buffer or sleeves; hence, r_1 becomes simply the radius of the pin.

(1) Riser-extension Joint at the Clevis

Equation (4-1) yields the allowable load for the riser extension:

allow load =
$$\frac{(15,512 \text{ lb}) \times (6 \text{ ply})}{2.82}$$
 = 33,000 lb. (4-1s)

The margin of safety, from use of Eq. (4-2), is calculated to be

M.S. =
$$\frac{33,000 \text{ lb}}{28,300 \text{ lb}} - 1 = +0.17$$
. (4-2s)

(2) Riser-extension Joint at the Payload Attachment .

The allowable load for the riser extension at the payload attachment is

allow load =
$$\frac{(15.512 \text{ lb}) \times (6 \text{ ply})}{2.68}$$
 = 57,800 lb. (4-lt)

The margin of safety is

M.S. =
$$\frac{34,800 \text{ lb}}{28,300 \text{ lb}} - 1 = +0.23$$
. (4-2t)

(3) Riser-extension Splice

(1) Margin of Safety on the Basis of Theory

Since the riser-extension splice is the same as that for the center line with the exception of the wrap, and since the theoretical calculations ignore the influence of the wrap, the margin of safety for the riser-extension splice is the same as that for the center line; therefore, reference is made to Section 4.g(3)(1 of this volume.

(2) Margin of Safety on the Basis of Tests

Appendix & shows that testing reveals the allowable load for the riser-extension splice to be

allow load =
$$\frac{(1 - 1) \times (2 \text{ ply})}{(1 + 1) \times (2 \text{ ply})} = 11,850 \text{ lb.} (4-1u)$$

TABLE 7 MATERIAL AND WEIGHTS

	Iton	Material	Qty, yd	Mt, 20
1.0	Canopy assembly			
1.1	Canopy cloth	36-in. nylon, 1.6 os/yd2, 0.G., MiL-C-7020, Ty II	2025	202.5
1.2	Skirt and vent reinf.	1 W nylon web, color opt., MIL-W-5625	200	16.3
	Rip tapes	1/2 W nylon tape, 0.D., MIL-T-5038, Ty III	99	9.0
7:	Susp. lines, pocket bands, reefing-ring		000	9
	10 に 日 に 日 に 日 に 日 に 日 に 日 に 日 に 日 に 日 に	Nylon cord, U.D., Hill-C-Judo, Ty III	200	3
1.5	Suspline reinf.	9/16 M nylon web, 0.D., MIL-W-4088, Ty I	52	*.0
1.6	Cutter patch	Nylon cloth, 7.25 oz/yd2, 0.D., MIL-C-7219, Ty III	-4	0.5
1.7	Reefing 11ne	Nylon cord, color opt., MIL-C-7515, Ty VII	20	5.0
7.8	Reefing-line wrap	3/4 N adhesive-back ripstop	*	•
1.9	Thread	Nylon, #1se E, O.D., V-T-295, Ty I, Cl I; and crtton, 5-cord, O.D., MIL-T-5660	1 1	•
SUET	suetotal			115.0
2.0	Center-line assembly			
2.1	Main web	1-3/4 nylon web. 0.D., MIL-W-27265, Ty XXVI	123	39.1
2.2	Vrap	1-3/4 nylon web, 0.D., MID-W-27265, Ty XII	2/3	
2.3	Spacer	Nylon duck, sage green, MilC-3953, Cl 2	2/3	
2.4	Binding	3/4 W nylon tape, C.D., MIL-T-5038, Ty III	ı	•
2.5	Thread	Nylon, size F. O.D., V-T-295, Ty I, Cl I, and Nylon, 6-cord, O.D., V-T-295, Ty I, Cl I	,	
SUBTOTAL	O'FAL			39.0

TABLE 7 con't

	المدودة والمستجددة والمستخددة والمستجدد والمستجدد والمستجدد والمستجدد والمستجدد والمستجدد والمستجدد والمستجدد والمستجدد والمستجد والمستجدد والمستجد والمستجدد والمستجد والمستجدد والمستجد والمستجدد والمستجد والمستجدد والمستجد والمستجدد والمستجد والمستجدد والمستجد والمستجدد والمستجد والمستجد والمستجد والمستجد والمستجد والمستجد والمستجد والمستجدد والمستجدد والمستجدد والمستجدد والمستجدد والمستجد والمستجدد والمستجدد والمستحد والمستح	والإسارة والمرازية		
3.0	Missr sesembly			
3.1	Main web	1 W mylen web, 0.D., MIL-W-27265, Ty XVIII	672	104.9
3.2	Laorda	1-3/4 W nylon web, 0.D., MIL-W-27265, Ty XIII	9	
3.3	dra.	1-3/4 W nylon web, 0.0. MIL-W-27265, Ty VIII	~	
3.4	Buffer	1-3/4 W cotton web, 0.D., MIL-W-5665, Ty XVII	1-1/2	}
3.5	Thread	Nylon, 6-cord, 0.D., V-T-295, Ty I, S1 I	ı	
SUBT	SUBTOTAL			106.4
4.0	Riser-extension assembly			
4.4	Main web	1-3/4 W nylon web, 0.D., MIL-W-27265, Ty XXVI, C1 R	270	82.5
	Spreader	1-3/4 W nylon web, 0.D., MIL-W-27265, Ty XIII, C1 R	٠,	6.1
.3	Spreader reinf.	1-3/4 W nylon web, 0.D., MIL-W-27265, Ty VIII, C1 R	:n	0.5
*:	Wrop	1-3/4 W nylon web, 0.D., MIL-W-27265, Ty VIII, C1 R	~	
* .S	Sliding keeper	1-3/4 W nylon web, 0.D., MIL-W-27265, Ty XIII, C1 R	~ +	
9.	Auffer	Cutton eleth, O.D., MIL-C-5645, Ty III	ı	
7:4	Thread	Mylon, 6-cord, 0.D., V-T-295, Ty I, Cl 1	:	ن ب
		Nylon, size E, O.D., V-T-295, Ty I, Cl l	1	
		Cotton, 6-cord, 0.D., Mil-T-5660, Ty II, Style A	,	
SUBT	SUBIOTAL			65.0
	Bag-bridle assembly			
5.1	Main web	1-3/4 cotton web, 0.D., MIL-M-5665, Ty X or XV, Cl 28 or 3	6	7.0
5.5	5.2 Wrap, buffer	1-3/4 cotton web, 0.D., MIL-W-5665, Ty VIII, Cl 28 or 3	1-1/3	
 	5.3 Thread	Nyion, 6-cord, O.D., V-T-295, Ty 1, Cl 1	i	5
SUBT	SUBTOTAL			4 64

TABLE 7

	• 4		
6.0 Deployment-bag			
SUBTOTAL	Refer XII-i-1654	-4	25.0
7.0 Hardware			25.0
7.1 Vent ring 7.2 Soparable link 7.3 Clevis assembly 7.4 Cutter brasket 7.5 Neefing ring SUBTOTAL	Nofer X11-1-1649 Refer M322002-1 Refer M370D67-7 and X11-1-1652 Refer 64D22262	30 m	0
TOTAL WEIGHT			18.5
OVERAGE ((569.7 1b) # 0.133	33		549.1
Total estimated weight of packed for service (total	Total estimated weight of one prototype parachute assembly packed for service (total weight last American total weight last		76.9
			-

The margin of safety is

M.S. =
$$\frac{11,850 \text{ lb}}{(28,300 \text{ lb})/3} - 1 = +0.25.$$
 (4-2u)

Once again, it is shown that the margin of safety for the splice derived from theoretical calculations is reasonably conservative. It can also be seen that use of a stronger wrap for the riser-extension splice yields a slightly higher margin of safety than for the splice for the center line using the lesser-strength wrap.

5. MATERIAL LIST AND WEIGHTS

The materials comprising the prototype parachute assembly are listed in Table 7 along with corresponding weights. The materials list accounts for approximately a 13% overage; this overage is deducted to arrive at a reasonable estimated total weight.

6. ACKNOWLEDGMENTS

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APPENDIX A

LABORATORY TEST REFORTS

E-0154, TL SERIES

FOREWORD

The tests reported in this Section were made to ascertain the structural integrity of the primary structural members of the 135-ft nominal diameter cargo parachute, NLABS drawing no. X11-1-1645.

TABLE A-1 LABORATORY TEST RUPORTS E-0154, TL SERIES

West No.	Itom(a) tostel	
~;	Mylon elotte contant and a contant	ତ ୍ତ ଶ
ť		₹ 0
V	When cloth, bias-cut, control sample	99
~)	Reeling line, control sample	, 13 (2)
্য	Mylon cord, control sample)) (
ũ	Mylon web, MIL-W-5625, 4,000 lb, control sample	5 6
ંગ	Hylon web, 1 W, MIL-W-27265, Ty, XVIII, control sample	· F
1.	Mylon web, 1 3/4 W, Mil-W-27265, Ty. XXVI, control sample	• 0
၁	Attachment, vent line to vent ring	, r
0-1	Attachment, vent line to vent ring	2 2
6	Attachment, vent line to vent band	
10	liem, vant band	- (
1.1	Joint (on the bias), main seam	2 3
12	Joint, cross seam	d (
13	ilem, skirt band	ນ .
		es S

(continued on next page)

TABLE A-1 (continued) LABORATORY TEST REPORTS E-0154, TL SERIES

7est Mo. 5-0154, 7L	Item(s) tested	agn _d
11	Attachment, suspension line to skirt	87
15	Attachment, suspension line to link	89
70	Attachment, reefing ring	16
17	Reefing line aplice	83
17-1	Reering line splice	95
18	Attachment, center line to vent ring	ა ა
10	Attachment, center line to clevis	86
02	Riser extension splice	100
77	Center line splice	102
8.5	Attachment, riser to clavis	103
~; ?i	Attachment, riser to link	105

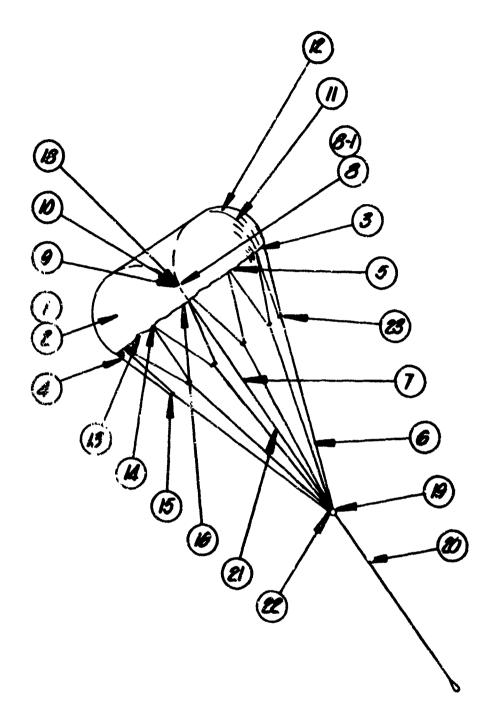
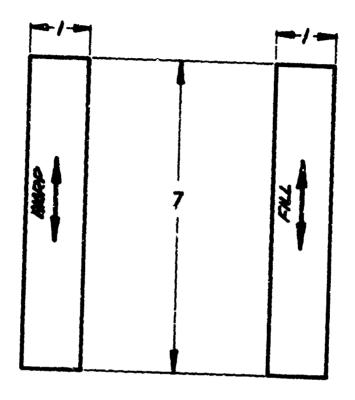


Fig. A-1. Key to laboratory-test reports, E-0154, TL series for 135-ft Nom. Dis. cargo parachute, N/LABS Deg. No. X11-1-1644.

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LABORATORY TEST REQUEST/REPORT

	Sylan Cloth, control sample					
Nylon Cloth, control sample MIL-C-7020, Ty. II NO. TL/1						
PURPOSE QUETIMATE OPOINT OF DEFFICIENCY OTHER STRENGTH FAILURE						
TEST METHOD Test in accordance with Federal Specification CCC-T-191b, method 5104. Use Scott Tester, Model J-3, 110-1b capacity with 12 in/min load rate.						
REQUESTED BY DATE REQUESTED REQUEST APPD. BY DATE APPROVED 11/21/68 RAT 11/21/68						
Ult. Strength, 1b/in. Sample Warp Fill						
1 2 3 4 5	63 79 69 76 71 74 70 76 71 75 69 76					
RESULTS All failures occurred over minimum ultimate rated strength.						
CONCLUSIONS						
TESTED BY	. Bivere 11/22/	SEATE	COMPLETED			

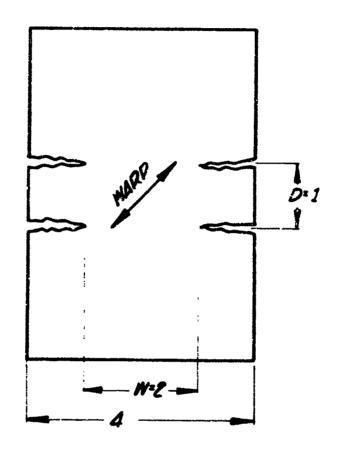


NOTE: SPECIMANS THREADED TO ABOVE DIMENSIONS

MYLON CLOTH CONTROL SAMPLE

SKETCH E-OISA, 72/1

ITEM(S) TO BE TESTED				PROJECT NO.	E-0154	
			itrol sample II Bias-Cut		TEST NO.	TL/2
PURPOSE	STRENGTH FAILURE					
TEST METHOD Similar to Federal Specification CCC-T-191b, Method 5100, except test 5 samples cut on the bias. Use Tinius Olsen Testing Machine, 500 lb scale, with 12 in/min load rate.						
REQUESTE MMK	D BY	DAT	E REQUESTED 11/15/68	'	T APPD. BY AT	DATE APPROVED 11/15/68
Sample 1 2 3 4 5 Av.	124. 124. 173. 129. 131.	8 2 0 6	Ult. bias strength, lb/in 124.8 124.2 173.0 129.6 131.4 136.6	is c foll Ult. wher stre W is test	Ultimate be alculated owing relations = P of the next of the original specimen.	tienship:
TESTED B	Y		Enor 11/16/	cai()87 =	COMPLETED	
	<u></u>	.D.	apor 11/10/	ce:	4000 60 160	



NYLON CLOTH, BIES-CUT CONTROL SAMPLE

SKETCH E-O154, TL/Z

ITEM(S) TO BE	TESTED		PROJECT	r	}	
Reafing Idea			NO.		E-0154	
Reefing Line, control sample Spec. MIL-C-7575, Ty. VII			NC.		TL/3	
PURPOSE ULTIMATE POINT OF EFFICIENCY OTHER STRENGTH FAILURE						
TEST METHOD Same as Wederal Specification CCC-T-191b, Method 4102, except Test 5 samples and report to the nearest 5 pounds. Use Tinius Olsen Testing Machine 12,000 lb capacity with 12 in/min load rate.						
REQUESTED BY	DATE REQUESTED	REQUES	T APPD.	BY	DATE APPROVED	
MMK	10/16/68		RAT		10/16/68	
TABLE		COMM	ENTS			
Sample Ult	. Strength, lb.	_				
1 2 3 4 5	2755 2680 2800 2730 2780 2749					
strength.	failures occur	red ove	er min.	ult	imate rated	
CONCLUSIONS						
TESTED BY K.	Hinkle 10/17/6	8 DATE	COMPLE	TED	10/18/68	

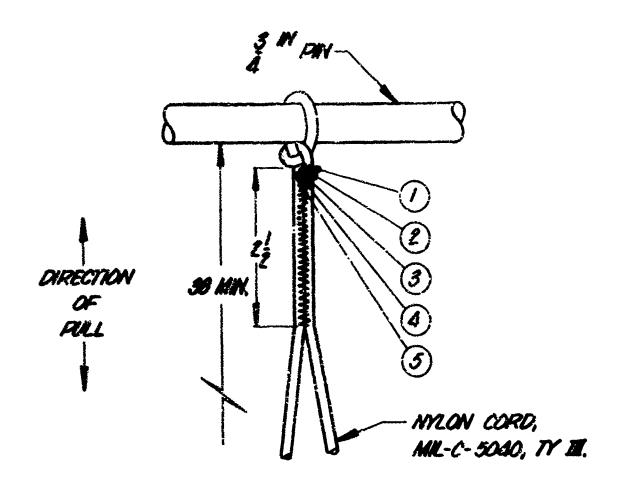
ITEM(S) TO BE TESTED Nylon Cord, control sample			PROJECT NO.	E-0154		
MIL-C-5040,	Ty. III		TEST NO.	TL/4		
PURPOSE ULTIMATE POINT OF DEFFICIENCY OTHER STRENGTH FAILURE						
TEST METHOD Test in accordance with Pederal Specification CCC-T-191b, Method 4102. Use Tinius Olsen Testing Machine 2400 1b capacity with 12 in/min load rate.						
REQUESTED BY	DATE REQUESTED 11/21/68		T APPD. BY AT	DATE APPROVED 11/21/68		
TABLE		COMM	ENTS			
Sample Ul	t. Strength, 1b	<u>.</u>				
1 2 3 4 5	640 575 582 580 580					
Av.	591					
RESULTS All strength.	failures occur	red ove	er minimum	ultimate rated		
CONCLUSIONS	CONCLUSIONS					
TESTED BY La	Rivere 11/25/6	8 DATE	COMPLETED			

ITEM(S) TO B	E TESTED		PROJECT	E-0154	
Nylon Web	control sample		TEST		
MIL-W-5625,	1 W 4000 1b t.	5.	NO.	TL/5	
PURPOSE WULTIMATE POINT OF STRENGTH FAILURE					
TEST METHOD Test in accordance with Federal Specifica-					
tion CCC-T-191b, Method 5100. Use Tinius Olsen Testing Machine, 12000 lb capacity, with 4 in/min load rate.					
REQUESTED BY	DATE REQUESTED	REQUES	T APPD. BY	DATE APPROVED	
MMK	11/21/68		RAT	11/21/68	
TABLE		СОММ	ENTS		
Comple III				į	
Sample Ul	t. strength, 1b				
1	4710 4600	i i			
2 3 4	4550	1			
	4520	i			
5	465.0	}			
Av.	4598				
		1			
RESULTS All	specimens fail	ed abov	ve min. rat	ed strength.	
	•				
CONCLLETONS					
CONCLUSIONS					
TESTED BY	a Rivere 11/25/6	DATE	COMPLETED		
	a. ロレメミンエーナフィング/ [

ITEM(S) TO BE			PROJECT		
Nylon Web,	control sample 7265, Ty. XVIII		NO.	E-0154	
C1 R, 6000	C1 R, 6000 1b t.s. NO. TL/6				
PURPOSE DUL ST	TIMATE POINT FAILU		EF71C1ENCY	OTHER	
Test in accordance with Federal Specification CCC-T-191b, Method 5100. Use Tinius Olsen Testing Machine, 12,000 1b capacity, with 4 in/min load rate.					
REQUESTED BY	DATE REQUESTED 11/4/68	i -	T APPD. BY	DATE APPROVED 11/4/68	
TABLE		СОММ	ENTS		
Sample Ul	t. strength, 1	<u> </u>			
1	7250				
2 3	7259 7251				
Av.	7253				
		1			
		1			
٠					
		1			
RESULTS A1	l specimens fail	ed abo	ve the min	imum rated	
strength.		455	VO 0110		
CONCLUSIONS					
TESTED BY L	a Riviere 11/5/	68 DATE	COMPLETED		
					

ITEM(S) TO BE	TESTED	PROJECT				
	control sample	NO.	E-0154			
1 3/6 W, MIL-W-27265, Ty. XXVI, TEST NO. TI/?						
	TIMATE POINT RENGTH FAILU		NCY OTHER			
CCC-T-191b,	TEST METHOD Test in accordance with Pederal Specification CCC-T-191b, Method 5100. Use Tinius Olsen Testing Machine 60,000 lb cap					
REQUESTED BY	DATE REQUESTED 11/4/68	REQUEST APPD. RAT	BY DATE APPROVED 11/4/68			
TABLE		COMMENTS				
Sample Ul	t. strength, 1b					
1 2 3	15,500 15,520 15,517					
AV.	15,512	6.74 6.74				
RESULTS All strength.	specimens faile	d above minimu	m rated			
CONCLUSIONS						
TESTED BY La	Riviere 11/5/68	DATE COMPLET	ED			

Attachment, vent line to vent ring PROJECT NO. E-0154 TEST NO. TL/8 PURPOSE ULTIMATE POINT OF EFFICIENCY OTHER STRENGTH FAILURE TEST METHOD Use Tinius Olsen Testing Machine, 2,400 lb capacity, with 12 in/min load rate. Pabricate and test 5 samples per attached sketch. Report results to					
the neares					
	DATE REQUESTED	·			
HOCK	10/18/68	R	AT	10/18/68	
1 2 3 4 5 Av.	1t. strength, 1b 658 752 706 722 740 716	fail	ee sketch i	for some of	
conclusions	efficiency is the knot end rep	the jo	rd) = 100 (than expected	
TESTED BY K.	Hinkle 10/18/68	DATE	COMPLETED		

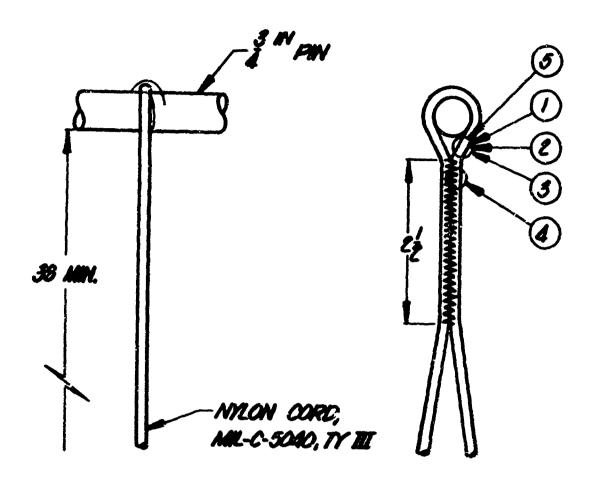


NOTE: ZIG-ZAG STITCHING SHALL BE 7-IL ST/IN COUNTING ON THE SIDE ROW WITH NYLON SIZE "E" THREAD USING A 2 STEP MACHINE. STITCHING SHALL BE 48 : " MIDE.

LINE ATTACHMENT TO VENT RING

SKETCH E-OKSA, TL/B

ITEM(S) TO BE			PROJECT				
	vent line to ve		NO.	R-0154			
ring. Ref. X11-1-1646,	NALABS Deg. no. Detail K	•	NO.	TL/8-1			
PURPOSE QUITIMATE QPOINT OF DEFFICIENCY OTHER STRENGTH FAILURE							
TEST METHOD	Use Tinius Olsa	n Test	ing Machin	e with 32			
in/min load attached ske pound.	Use Tinius Olsen Testing Machine with 12 in/min load rate. Fabricate 5 samples and test per attached sketch. Report the results to the nearest pound.						
REQUESTED BY	DATE REQUESTED	REQUES	T APPD. BY	DATE APPROVED			
MIK	10/18/63	<u> </u>	RAT	10/18/68			
YABLE		СОММ	ENTS				
Sample Ult	strength, lb		e attached of failur	sketch for			
1	1,050						
2	996	1					
3	1,048						
5	1,032 998	1					
1	730	1					
Av.	1,025	1					
İ		ļ					
1							
1		}					
		l					
RESULTS							
strength of	Miciency of Join Joint/Av. ult.	t is: streng	100 × (Av th of cord	. ult.) = 100			
(1025/1182)	- 0/3.						
CONCLUSIONS							
	Ultimate stre or intended app	ength o clicati	r the atta on.	chment 1s			
TESTED BY R.	. Hinkle 10/18/6	8 DATE	COMPLETED)			

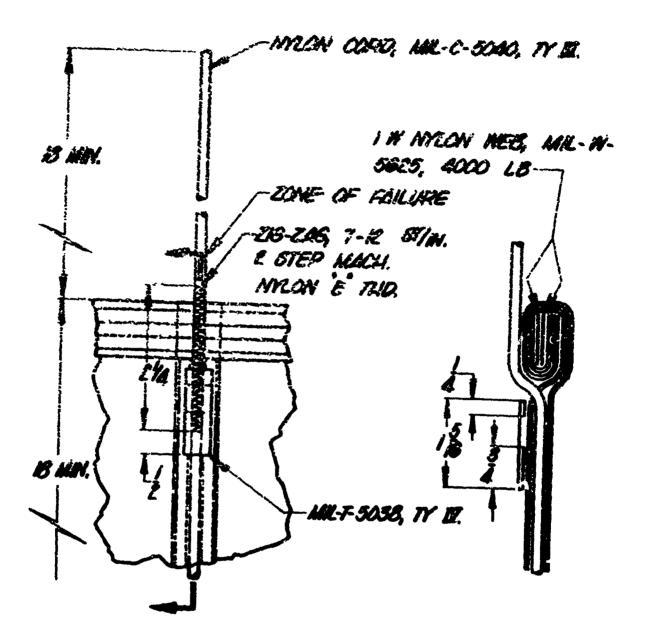


NOTE: ZIG-ZAG STITCHING SHALL BE 7-IL ST/IN COUNTING ON THE SIDE ROW MITH NYLON SIZE E THREAD USING A 2 STEP MACHINE. STITCHING SHALL BE 18 : 05 MIDE.

LINE ATTACHMENT TO VENT RING

SKETCH E-OKSA, TL/B-1

ITEM(S) TO BE TESTED			PROJECT E-0154		
Attachment, ven	stachment, vent line to went			E-0154	
band. Ref. NAL				TL/9	
PURPOSE DULTIMUSTREM	ATE POINT	OF E	efficiency	OTHER	
TEST METHOD SAM	e as Federa	Speci	fication C	CC-T-191b	
Method 4102 exc	ept fabrical	te 5 88	sples and	oull per	
Method 4102 exc attached sketch	. Use Tinta	18 01se	n Testing	Machine,	
2,400 lb capaci	ty with 12 t	ln/min	load metr	and report	
to the nearest	ponug.				
REQUESTED BY DAT	E REQUESTED	REQUES	T APPD. ST	DATE APPROYED	
	10/2/68	R	AT	10/3/68	
TABLE		COMM	ENTS		
Sample Ult. 8	trength, 1b	S fail		for zona of	
1	636				
2	638	[
1 2 3	620 633	1			
5	632 640	1			
,	040	1			
Av.	633				
		1			
		i			
RESULTS					
	efficiency i	ls: 10	0 × (Av. u	lt. strength	
of joint/Av. ul	t. strength	of cor	a) = 100 (t	633/591 lb) =	
1078.					
					
CONCLUSIONS D	ltimate stre	ngth o	f the joins	t is acceptable	
for intended ap	plication.		—	 	
TESTED BY M. M.	Knor 10/9/68	DATE	COMPLETED		
A. D. W. WHOT. TALLAGO TOWIE COMPETED					

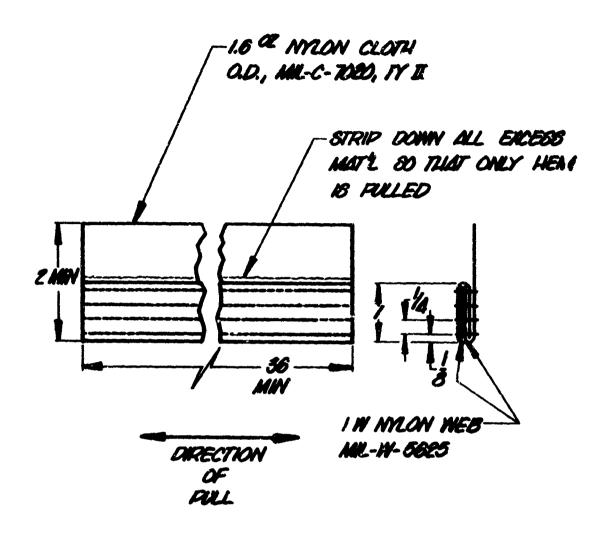


NOTE: MACHINE STITCHING UNLESS OTHERWISE SPECIFIED SHALL BE TYPE 301, FED STD TSI 8-IL STIN WITH NYLON SIZE "E" THO.

VENT LINE ATTACHMENT

EXETCH E-0154, TL/10

ITEM(S) TO BE TESTED No. 10. B-0154 PURPOSE ULTIMATE POINT OF FEFFICIENCY OTHER STRENGTH FAILURE TEST Method Use Tinius Olsen Testing Nachine, 12,000 1b								
federal Spe fabricate 5	repacity, with 12 in/min lead rate. Test same as Federal Specification CCC-T-191b, Method 4102, except fabricate 5 samples and pull per attached sketch. REQUESTED BY DATE REQUESTED REQUEST APPD. BY DATE APPROVED							
1 2 3 4 5	8580° 8540°° 8540°° 8580 8220 8550°°°	80 PM	eloeded to 5400 lb eloaded to 4950 lb eloaded to 6500 lb					
RESULTS OF JOIND/(AV 9196) = 935.	ciency of joint	is: 100 of webbir	0 = Av. ult strengthing = 2) = 100 (8514/					
fer intende	d application.	- 	ent band is acceptable					
TESTED BY N.N	. Knor 10/9/68	DATE CO	OMPLETED					

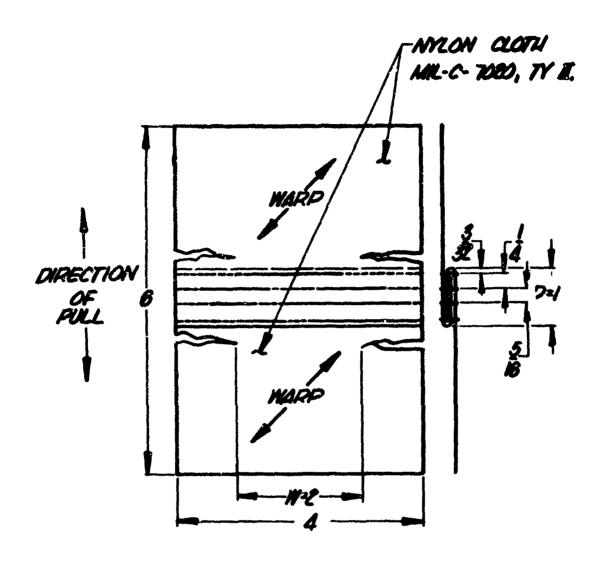


NOTE: MACHINE STITCHING SHALL BE TYPE SON FED STD TSI 8-11 ST/M WITH MYLON SIZE E

HEM, WENT BAND

SKETCH E-0154, TL/10

Joint (on the bias), main seam				ROJECT NO.	F_016h
Ref.	NALABS	Dwg. No. X11-1	-1645		E-0154
Detail A				NO.	TL/11
PURPOSE	STRE	IMATE POINT		JEFFICIENCY	OTHER
TEST ME	THOD S	ame as Pederal	Speci	fication Co	CC-T-191b.
attaci 600 li	d 5100, hed ske b capac	except fabric tch. Use Tini ity with 12 in	ate and lus Olso n/min lo	d test 6 sa en Testing oad rate.	amples per Machine,
REQUEST MMK		ATE REQUESTED 11/18/68		T APPD. BY RAT	DATE APPROVED 11/18/68
TABLE			СОММ	ENTS	
Sample 1 2 3 4 5 6	F, 16 128.0 125.0 119.0 146.0 119.0	128.0 125.0 119.0 167.0 146.0 119.0	l. is followher strew is specific to the strew	Ultimate is calculated lowing related bias streemen F is the ength of the original cimen, in.	ength = $\frac{P}{W-D}$,
 RESULTS Efficiency of the joint is: 100 × (Av. ult. bias strength of joint/Av. ult. bias strength of cloth) = 100 (134/136.6) = 98%. Efficiency of the joint is: 100 × (min. ult. bias strength of joint/min ult. bias strength of cloth) = 100 (119.0/124.2) = 95.8%. 					
CONCLUSIONS Ultimate strength of main seam is acceptable for intended application.					
TESTED	BY M. M.	Knor 11/20/6	8 DATE	COMPLETED	



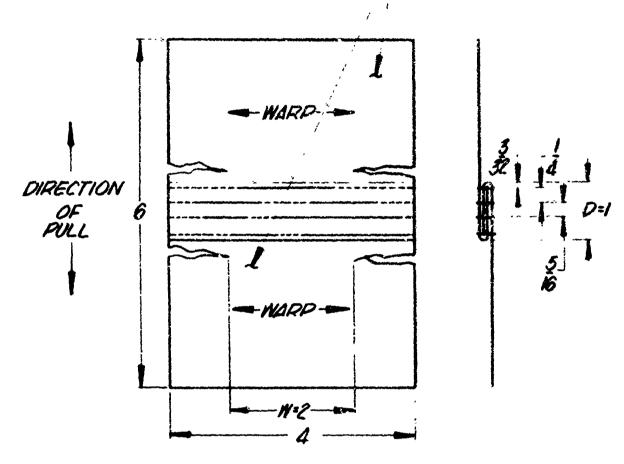
NOTE: MACHINE STITCHING SHALL BE TYPE SOI FED STD 751 8-11 ST/IN WITH MYLON SIZE E

JOINT (ON THE BIAS), MAIN SEAM

SKETCH E-OKS4, TL/N

ITEM(S) TO B	E YESTED		PROJECT			
			NO. R-0154			
Joint, Cros		3665	TEST			
	Dwg. No. X11-1-	·1042,	NO.	TL/12		
PURPOSE TO U	TIMATE TPOINT	OF G	EFFICIENCY	COTHER		
	STRENGTH FAILURE					
TEST METHOD Same as Federal Specification CCC-T-191b, Nethod 5100, except to be fabricated and tested per						
Method 5100	, except to be	fabrica	ted and ter	sted per		
attached sk	etch. Use Tinio	25 Olse	n Testing !	Machine,		
600 lb scal	e with 12 in/mir	1 load	rate.			
AFALICATED BY	Toate acquiertes	lacoure.	F 4880 5V	DATE 40000155		
	DATE REQUESTED	l '				
MMK	11/15/68	H	AT	11/15/68		
TABLE		COMM	ENTS			
Gammia 171	4 strongth 1h		ince test	section is 2		
Sample U1	t. strength, lb	in.		PACUTON TE C		
1	121.0					
2	111.0	د ا	Y			
3	129.0	1 =	2 * AV. 8	trength for		
á	119.0	1	in. secti	on of seam.		
2 34 56	117.0	1 -				
6	123.0	i				
		1				
Av.	120.0	l				
		i				
For 2 in.	test section					
		1				
		ĺ				
		i				
RESULTS			A) - A			
	verage ultimate					
crotu in th	e fill direction	n 3.5 /0	10/1R (Se	e test - TW/1).		
	fficiency of jo	int ie	160 = /4	v. ult.		
strength of	joint/Av. ult.	strens	th of clok	h) =		
100 (60/76)		00.000	, 01. 01. 01.	•••		
,, ,-,	1287					
1						
Ī						
CONCLUSIONS						
1	Iltimate strengt	h of ci	oss seam 1	s acceptable		
	d application.					
	*					
TESTED BY M.	M. Knor 11/20/	68 DATE	COMPLETED			
TESTED BY N. M. Knor 11/20/68 DATE COMPLETED						

MIL-C-7020, TY II.

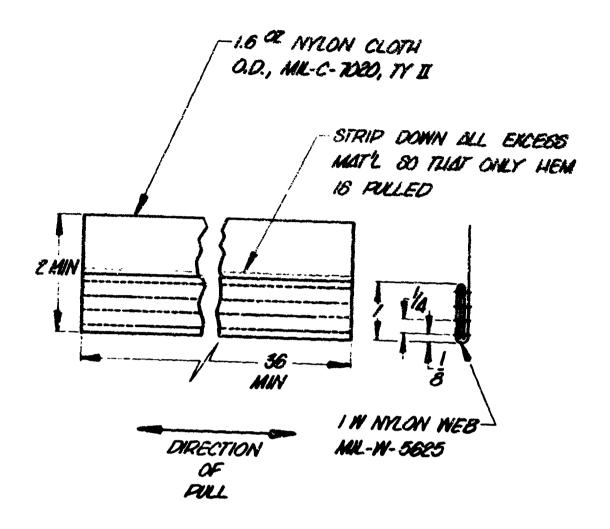


NOTE: MACHINE STITCHING SHALL BE TYPE 301 FED STD 751 8-11 STIN WITH NYLON SIZE E

JOINT, CROSS SEAM

SKETCH E-0154, TL/18

ITEM(S) TO BE Hem. skirt			PROJECT NO.	E-0154
Ref. NALABS Detail B	-1645,	TEST NO.	TL/13	
PURPOSE W UL	TIMATE POINT		EFFICIENCY	OTHER
TEST METHOD	Use Tinius Olse	n Test	ing Machine	
Federal Spe	th 12 in/min lo cification CCC- samples and to	T-191b	, Method 41	102, except
REQUESTED BY	DATE REQUESTED	REQUES	T APPD. BY	l
MMK	10/2/68	R	AT	10/3/68
TABLE		COMM	ENTS	
Sample Ul	t. strength, 11		Repeated to Preloaded	
1 2	4620* 4475**			
2 3 4	468G	Ì		
4	4530			
5	4565			
Av.	4574	Ì		
		İ		
RESULTS				
of joint/A	iciency of joins v. ult. strength	t 1s: n of we	p) = 100 (ult. strength 4574/4598) =
995.			•	•
CONCLUSIONS Ultimate strength of skirt band is				
acceptable for application intended.				
TESTED BY M.	M. Knor 10/4/6	8 DATE	COMPLETED	

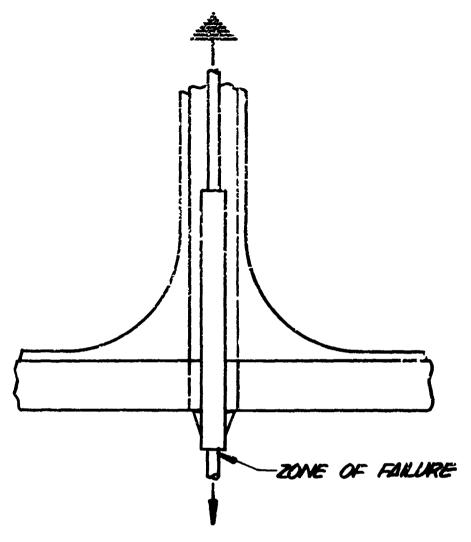


NOTE: MACHINE STITCHING SHALL BE TYPE 301 FED STD 751 8-11 ST/W WITH NYLON SIZE E"

HEM, SXIRT BAND

SKETCH E-0154, TL/13

ITEM(S) TO BE	FTESTED		PROJECT	
			NO.	E-0154
Attachment, suspension line to		to	TEST	
skirt. Ref. NALABS Dwg. no. X11-1-1645, Detail B			NO.	TL/14
	TIMATE POINT	05 F	EEE161ENCY	COTHE
	RENGTH FAILU		EFF161ENCY	Polyex
TEST METHOD	Same as Pederal	Speci	fication C	"C-T-1615
Method 4102.	except fabrics	ite 5 s	amples and	test per
attached ske	etch. Use Tiniu	s Olse	n Testing I	fachine,
2400 lb caps	city with 12 in	/min l	oad rate.	·
REQUESTED BY	DATE REQUESTED	REQUES	T APPO BY	DATE APPROVED
MMK	10/2/68	1	RAT	
LE-IV	10/2/68		DA1	10/3/68
TABLE		COMM	ENTS	
Sample Ult	. strength, 1b			sketch for
	***	zon	e of failu	e.
1 1	600 584	1		
2	600	j		
3	576			
5	592	1		
1	JJ#	1		
Av.	590	- 1		
;	• •	1		
Ì		1		
		1		
1		1		
		l l		
RESULTS	indianta that	dadma	- 3005 -	Platant and
no loss of	n indicate that structural integ	. Jaint	f the suspe	neion line
occurred.	or accorded Times	,110,7 0	c cas suope	mozon zine
· I				
ł				
\				
1				
CONCLUSIONS				
Ultimate strength of the joint is acceptable				
for application intended.				
TESTED BY	M. Knor 10/9/6	PIDATE	COMPLETED	
<u> </u>	. M. KNOT 10/9/6	70]	30.00 22.720	



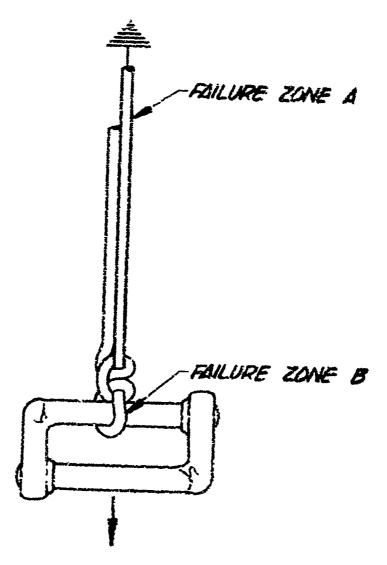
DIRECTION OF PULL

SEE DWG NO XII-I-1645 FOR DETAILS OF MATERIALS AND JOHNNS.

ATTACHMENT, SUSPENSION LINE TO SKIRT

SKETCH E-0154, TL/14

	O BE TESTED		PROJECT NO.	F.015*
Attachment, suspension line to link. Ref. NALABS Dwg. no. X11-1-1648			TEST NO.	E-0154 TL/15
PURPOSE [PURPOSE WILTIMATE POINT OF REFFICIENCY OTHER STRENGTH FAILURE			
TEST METH	OD Use Tinius Olse	n Task	ing Montain	2800 35
and test	with 12 in/min loa as per attached sk	d rate.	. Papricat	e 5 samples
REQUESTED	BY DATE REQUESTED	REQUES	T APPD. BY	DATE APPROVED
MMK	10/21/68	1	RAT	10/21/68
TABLE		COMM	ENTS	
Sample	Ult. strength, 1b		ee attached of failure	i sketch for
1 2 3 4 5	592 558 588 558 574	Safaile	amples 1, 3 ed in zone	3, 4 and 5
AV.	574			
RESULTS	Efficiency of tot		330 - / 4.	1174 Sharash
of joint.	Efficiency of joi /Av. ult. strength	nt 15: pî cor	130 * (A1 1) n 100 (5	7. uit. atrengt 174/551) = 973.
1	CONCLUSIONS Ultimate strength of joint is acceptable for intended application.			
TESTED BY	K. Kinkle 10/22/6	B DATE	COMPLETED	-

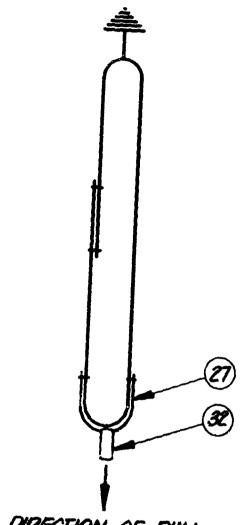


DIRECTION OF PULL

SEE DING NO XII-I-1648 FOR DETAILS OF MATERIALS AND JOINING.

SKETCH E-OISA, TE/15

Attrobuent mosting nine Post NO. E-0154				
Attachment reefing ring. Ref. NO. E-0154 NALABS Dwg. no. X11-1-1646, TEST			E-01)4	
Detail H. NO. TL/16				TL/16
PURPOSE TO UL	TIMATE POINT	OF [EFFICIENCY	OTHER
	RENGTH FAILU	KE	,, <u>-</u> -,	
TEST METHOD	Use Tinius Olse	n Test	ing Machine	600 1b
capacity wit	h 12 in/min los	d rate	. Fabricat	e 5 samples
	per attached sk			
	DATE REQUESTED			
MMX.	11/8/68	L	AT	11/8/68
TABLE		COMM	ENTS	
Sample Ult	. strength, 1b	•	Indicates ;	aw break.
1	405	ł		į
2 3 4	404	-		
3 h	378 203 *	1		
5	425			
Av.	403			
RESULTS Reefing ring pt. no. 48A7995 failed during				
each test.				
CONCLUSIONS				
Ultimate strength of attachment is acceptable for application intended.				
TESTED BY H. M. Knor 11/14/68 DATE COMPLETED				

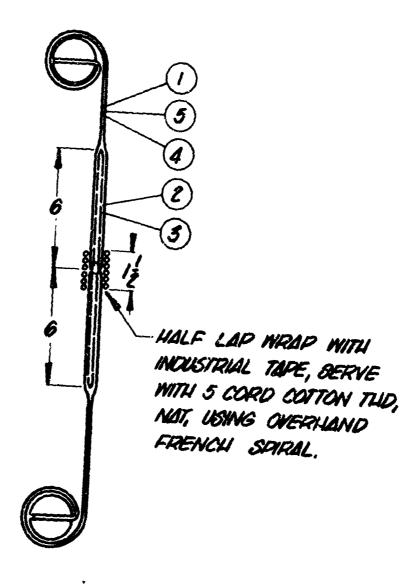


DIRECTION OF PULL

SEE DING NO XII-1-1646 FOR DETAILS OF MATERIALS

E-CUSA, TL/16

ITEM(S) TO BE	TESTED		PROJECT	
Joint meaf	ng line enlice		NO. TEST	E-0154
Joint, reefing line splice			NO.	TL/17
	TIMATE POINT FAILU		EFFICIENCY	OTHER
TEST METHOD	3.433 4.4			
Testing Mach	Build and test sine, 12,000 lb	capacit	ty with 12	in/min load
REQUESTED BY	DATE REQUESTED	REQUES	T APPD. BY	DATE APPROVED
MMK	10/17/68	1	RAT	10/17/68
TABLE		СОММ	ENTS	
Sample Ult	. strength, lb			
1	5610	1		
2	5670	1		
2 3 4	5680 5680			
5	5760			
•	5690	ĺ		
Av.	5680			
		1		
		1		
		1		
RESULTS		<u> </u>		
Average ultimate strength of control sample is 2,749 lb (see test TL/3).				
Efficiency of the joint is: 100 * (Av. ult. strength of joint/Av. ult. strength of cord) = 100 (5680/5498) = 103.31.				
) •				
CONCLUSIONS FIFT CAPPAR AS PRINTING PRINTING				
Efficiency is relatively high. Retest using different test set up.				
TESTED BY K.	Hinkle 10/17/68	DATE	COMPLETED	10/21/68
				20/ 61/ 00

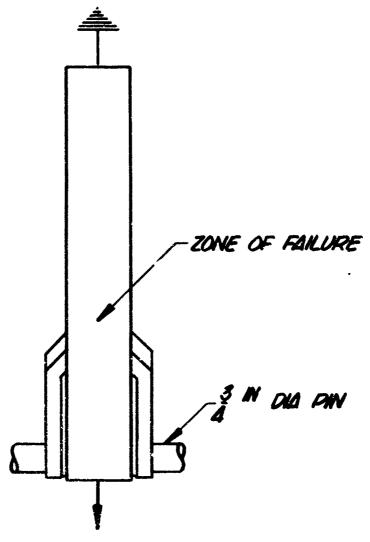


JOINT, REEFING LINE SPLICE

SKETCH E-0154, TL/17-1

ITEM(S) TO B	E TESTED		PROJECT	
		NO.	E-0154	
Joint, reef	Joint, reefing line splice		TEST NO.	TL/1/-1
PURPOSE IN S	LTIMATE E POINT TRENGTH FAILU		EFFICIENCY	OTHER
TEST METHOD	Build and test	6 comp	les es par	etteched
sketch. Us	e Tinius Olsen T			
capacity wi	th 12 in/min los	d rate	•	
REQUESTED BY	DATE REQUESTED	REQUES	T APPD. BY	DATE APPROVED
MMK	11/20/68	l .	RAT	11/20/68
TABLE		Гсомм	المستحدد المحد	
		50,31	,	
Sample Ul	t. strength, 1b	S		for zone of
1	2640			
2	2840 2550	A	verage ult:	imate strength ple is 2749 lb
3	2580		test TL/3	
5	2600			
Av.	2642			
		l l		
		- 1		
		-		
RESULTS				
	. Efficiency of the of joint/Av.			
	00 (2642/2749)			
و	. Ffficiency of	f the i	oint is:	100 × (Min.
2. Efficiency of the joint is: 100 * (Min. t.s. of joint/Min. t.s. of control sample) = 100 (2550/2680) = 95%.				
				İ
CONCLUSIONS	CONCLUSIONS			
Ultimate strength of splice is acceptable for intended application.				
TESTED BY M.	M. Knor 11/21/68	B DATE	COMPLETED	

ITEM(S) TO BE		want	PROJECT NO.	E-0154
ring. Ref. X11-1-1650	center line to NALAHS Dwg. no.	venc	TEST NO.	TL/18
PURPOSE TUL	TIMATE PO.	FX	EFFICIENCY	OTHER
TEST METHOD	Use Tinius Ola	en Tes	ting Machin	e. 60,000 lb
capacity with and test as	th 4 in/min load per attached si	i rate.	Fabricate	4 samples
	DATE REQUESTED	1		
MAK	12/12/68	H	TA	12/12/68
TABLE		COMM	ENTS	
Sample Uli	t. strength, lh	\$	See attache zone of fai	ed sketch for Llure.
1 2	27,050 27,300			
3	24,600			
4	24,600			
Av.	25,887.5			
RESULTS	iniancy of the	inint 1	** 100 ×	(Av. ult.
Efficiency of the joint is: 100 × (Av. ult. strength of joint/Av. ult. strength of control sample) = 100 (25,887.5/31,024) = 83.4%.				
CONCLUSIONS Ultimate strength of joint is acceptable for intended application.				
TESTED BY M.M	Knor 12/23/68	DATE	COMPLETED	



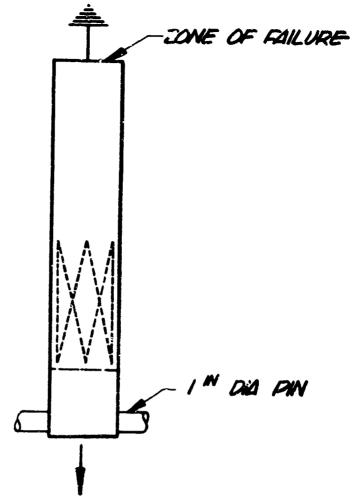
DIRECTION OF PULL

SEE DWG NO XII-1-1650 FOR DETAILS OF MATERIALS AND JOINING.

ATTACHMENT, CENTER LINE TO VENT RING

SKETCH E-OISA, TL/IB

ITEMES TO SE	r treven		000 15 15	· · · · · · · · · · · · · · · · · · ·
ITEM(S) TO BE Attachment.	center line to	clevis	PROJECT NO.	E-0154
Ref. NALABS	Dwg. no. X11-1-	1650	TEST NO.	TL/19
	TIMATE POINT	OF I	EFFICIENCY	
	STRENGTH FAILURE			
TEST METHOD	Use Tinius Olse	n Test:	ing Machine	, 60,000 lb
capacity wit	th 4 in/min load per attached sk	rate. metch.	Fabricate	e 3 samples
	DATE REQUESTED	1	1	
MMK	12/12/68	R	AT	12/12/68
TABLE		COMM	ENTS	
Sample Uli	t. strength, lo			i sketch for
1	28,800	the	zone of fa	llure.
2	28,450 28,000		uring each	test jaw at the upper
	7	fixt		at the upper
l Av.	28,419	1		
		ļ		
		1		
		1		
RESULTS				
Dat	ta obtained duri	ing the	test indi	cate that
can be assur	s 93% efficient. med that efficie	. Owin ency is	g to the jactually	better th an
indicated a	bove.			
CONCLUSIONS				
	Attachment is	accept	able for a	pplication
intended.				
TESTED BY M.	M. Knor 12/23/6	8 DATE	COMPLETED	



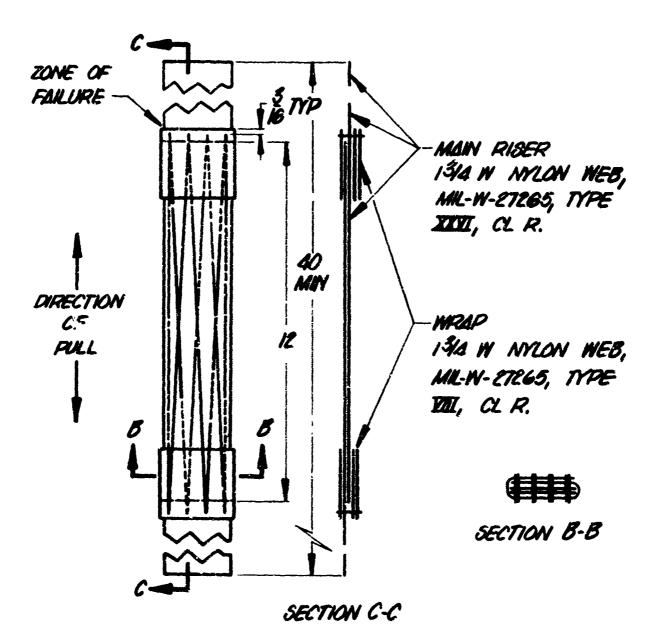
DIRECTION OF PULL

SEE DING NO XII-1-1646 FOR DETAILS OF MATLS

ATTACHMENT, CENTERLINE TO CLEVIS

SKETCH E-0154, TL/19

ITEM(S) TO BE	ETESTED	PROJECT		
Riser exter	sion splice	NO. E-0154		
		NO. TL/20		
PURPOSE DUL	PURPOSE WILLIAMS POINT OF BEFFICIENCY OTHER STRENGTH FAILURE			
TEST METHOD	Hea Tintue Old	sen Testing Machine 60,000 lb		
capacity wi	th 4 in/min los	ad rate. Pabricate and test		
	as per attached			
REQUESTED BY	DATE REQUESTED	REQUEST APPD. BY DATE APPROVED		
MKK	12/12/68	RAT 12/12/68		
ากก	12/12/00	10.7 12.7 00		
TABLE		COMMENTS		
		1		
Sample Ul	t. strength, 1b	.		
•	3.2.250			
3 4	13,350 13,500			
3	13,700			
4	13,550			
5	13,600			
۸۷.	13,540			
		į.		
RESULTS				
	t Affiniannu in	: 100 × (Av. ult. strength		
of joint/A	v. ult. strengt	h of control sample) =		
100 (13.54	0/15,512) = 87%	· · · · · · · · · · · · · · · · · · ·		
CONCLUSIONS				
CONCLUSIONS	Ultimate stre	ength of splice is acceptable		
for intend	ed application.			
TESTED BY M.	M. Knor 12/18/6	8 DATE COMPLETED		



MACHINE STITCHING TYPE 301, FED STD 751, 5 TO 8 STITCHES PER INCH WITH NYLON 6 OORD TUREAD.

RISER EXTENSION SPLICE

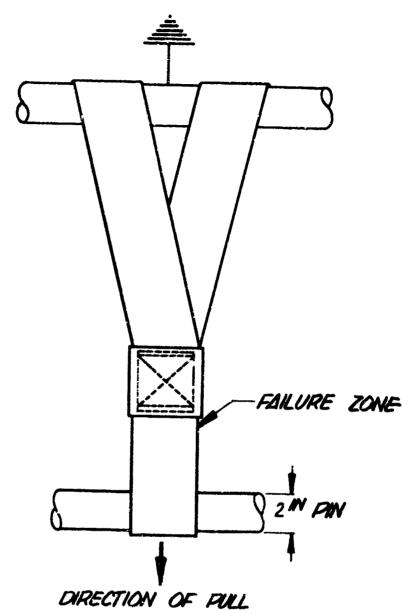
SKETCH E-0154, 12/20

LABORATORY TEST REQUEST/REPORT

ITEM(S) TO B	E TESTED		PROJECT	E-0154
Center-line	splice		TEST	
			NO.	TL/21
S	LTIMATE POINT TRENGTH FAILU	OF X	EFFICIENCY	OTHER
TEST METHOD	Use Tinius Cls	en Tes	ting Machin	ne, 60,000 lb
capacity wi 5 samples u	th 4 in/min loads per attached 3	rate. ketch.	Fabricat	e and test
	DATE REQUESTED			
MMK	12/12/68		RAT	12/12/68
TABLE		СОММ	ENTS	
Sample Ul	t. strength, lb			
1	13,800		ching fail	
2	13,900 12,200	Stit	below the ching fail	stitching. ed.
3	12,950	Stit	ching fail	ed.
5	13,450	3515	ching fail	ea.
Av.	13,400			
		Refe	r to sketc	h E-0154, TL/20
		and spli	note Fur c ce, wrap 1	enter-line s Ty. XII.
ſ		1	•	•
		┸		
RESULTS Joi	int efficiency is	s: 100	× (Av. ul	t. strength
of joint/Av	v. ult. strength	of con	trol sampl	e) =
100 (13,400	0/15,512) * 86\$.			
CONCLUSTONS	iiitimata etw	eneth o	f splice i	s acceptable
	Ultimate streed application.	ength o	f splice i	s acceptable

LABORATORY TEST REQUEST/REPORT

11EM(2) 10 B	E TESTED		PROJECT	_
Attachment	riser to clevis		NO.	E-0154
	Dwg. no. X11-1-		TEST NO.	TL/22
PURPOSE UL	TIMATE POINT		JEFFICIENCY	OTHER
TEST METHOD	Use Tinius Olse	n Test	ing Machine	. S0.000 1b
capacity with and test as	th 4 in/min load per attached sk	rate.	Pabricate	3 samples
REQUESTED BY	DATE REQUESTED	REQUES	T APPD. BY	DATE APPROVED
MMK	12/12/68	R	TA	12/12/68
TABLE		COMP	ENTS	
Sample Ul	t. strength, 1b		See sketch fallure.	for the zone
1	26,000			
2	24,400 26,150			
3	26,150			
Av.	25,517			
		1		
		1		
		1		
		1		
RESULTS			/ /	A
Joi	nt efficiency is ult. strength	5: 100 of com	ı × (AY. ül trol samol	t. strengtn e) =
100 (25,517	/28,012) = 883.	J1 601	oror ampt	~ /
	-			
1				
l				
			···	
CONCLUSIONS	Ultimate stre	ngth of	joint is	acceptable
for intende	d application.			
TESTED BY M.	M. Knor 12/23/	68 DATE	COMPLETED	
			····	



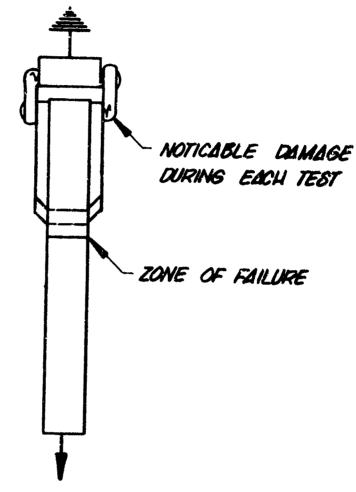
SEE DING NO XII-1-1651 FOR DETAILS OF MATERIALS

ATTACHMENT, RISER TO CLEVIS

S. ETCH E-0154, TL/EL

LABORATORY TEST REQUEST/REPORT

ITEM(S) TO BE TESTED	PROJ	- · ·
Attachment, riser to link	NO TEST	
Ref. NALABŠ Dwg. no. X11-1-1	651 NO.	
PURPOSE WULTIMATE POINT STRENGTH FAILUR	FEFFI	CIENCY OTHER
TEST METHOD Use Tinius Ol 12,000 lb capacity with 4 in	/min load	rate. Pabricate
5 samples and test as per at	tached ske	etch.
REQUESTED BY DATE REQUESTED F	EQUEST APP	D. BY DATE APPROVED 12/12/68
TABLE	COMMENTS	
Sample Ult. strength, 1b		ached sketch for of failure.
1 5,400	J 20116	v
2 5,460 3 5,520		
3 5,520 4 5,430		
5 5,250		
Av. 5,412		
RESULTS Joint efficiency is of joint/Av. ult. strength (100 (5,412/7.253) = 75\$.	: 100 × (f control	(Av. ult. strength sample) =
CONCLUSIONS Ultimate streng for intended application.	h of joint	t is acceptable
70000 m. M.M. Vara 10/10/60	,	
TESTED BY M.M. Knor 12/18/68	DATE COMP	LETED



DIRECTION OF PULL

SEE DNG NO XII-1-1657 FOR DETAILS OF MATERIALS AND JOINING.

ATTACHMENT, RISER TO LINK

SKETCH E-0154, TL/28

APPENDIX B
TRAJECTORY

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